



A Comparative Exergoeconomic Analysis of Waste Heat Recovery from a GT-MHR using Organic Rankine Cycles

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Contents

- 1) Introduction
- 2) Configurations of GT-MHR/ORC Combined Cycles
- 3) Exergoeconomic analysis (SPECO method)
 - 3-1) Identification and Analysis of Energy and Exergy Streams
 - 3-2) Defining the Fuel and Product for Each Component
 - 3-3) Cost Balances
- 4) Results and Discussion
 - 4-1) Exergoeconomic Analysis
 - 4-2) Parametric Study
- 5) Conclusions

1) Introduction

The world faces numerous sustainability challenges. Much of the world's energy is currently produced and consumed in ways that cannot be sustained. One approach to overcoming this problem is to develop and improve renewable energy sources. Another approach is to improve conventional energy converting systems so they efficiently utilize all the energy that can be extracted from a source. In the present work, methods for employing different configurations of ORCs for utilization of waste heat from the precooler of the GT-MHR are investigated from the exergoeconomic viewpoint.



The exergoeconomic analysis is performed based on the specific exergy costing (SPECO) approach.

The three considered ORC configurations are: Simple Organic Rankine Cycle (SORC), ORC with internal heat exchanger (HORC) and Regenerative Organic Rankine Cycle (RORC).

Also a parametric study is performed to reveal the effects of some important parameters on the exergoeconomic performance of the combined cycles.

2) Configurations of GT-MHR/ORC Combined Cycles



Turbine-Modular Helium Reactor/Simple Organic Rankine Cycle (GT-MHR/SORC)



Turbine-Modular Helium Reactor/ORC with internal heat exchanger (GT-MHR/HORC)



Turbine-Modular Helium Reactor/Regenerative Organic Rankine Cycle (GT-MHR/RORC)

3) Exergoeconomic Analysis

3-1) Identification and Analysis of Energy and Exergy Streams

$$\begin{split} \dot{Q} + \sum \dot{m}_{i}h_{i} &= \dot{W} + \sum \dot{m}_{e}h_{e} \\ \dot{E}_{Q} + \sum \dot{m}_{i}e_{i} &= \dot{E}_{W} + \sum \dot{m}_{e}e_{e} + \dot{E}_{D} \\ \dot{E} &= \dot{E}_{ph} + \dot{E}_{ch} \\ e_{ph} &= (h - h_{0}) - T_{0}(s - s_{0}) \\ e_{ch}^{mix} &= \left[\sum_{i=1}^{n} X_{i}e_{ch_{i}} + RT_{0}\sum_{i=1}^{n} X_{i}\ln(X_{i})\right] \\ e_{ch}^{mix} &= \left(\sum_{i=1}^{n} X_{i}e_{ch_{i}} + RT_{0}\sum_{i=1}^{n} X_{i}\ln(X_{i})\right) \\ \varepsilon &= \left(\sum_{i=1}^{n} X_{i}e_{ch_{i}} + RT_{0}\sum_{i=1}^{n} X_{i}\ln(X_{i})\right)$$

| Parameters | Value | | | | |
|---|---------|--|--|--|--|
| P_0 (kPa) | 100 | | | | |
| PR _C | 1.5-5 | | | | |
| $\dot{Q}_{\rm RC}$ (MW) | 600 | | | | |
| T_0 (°C) | 25 | | | | |
| T_1 (°C) | 700-900 | | | | |
| $T_{\rm C}$ (°C) | 40 | | | | |
| <i>T</i> _E (°C) | 80-120 | | | | |
| $\Delta T_{\rm E}$ (°C) | 2-10 | | | | |
| $\Delta T_{\rm Sup}$ (°C) | 0-15 | | | | |
| $\eta_{\rm P}(\%)$ | 85 | | | | |
| η_{T} (%) | 80 | | | | |
| Effectiveness (for IC, R, PC)(%) | 90 | | | | |
| $\Delta P_{\rm RC}$ (kPa) | 100 | | | | |
| $\Delta P_{\rm E}, \Delta P_{\rm IC}, \Delta P_{\rm PC} ({\rm kPa})$ | 40 | | | | |
| $\Delta P_{\rm R,HP}$ (kPa) | 80 | | | | |
| $\Delta P_{\rm R,LP}$ (kPa) | 50 | | | | |

 Table 1. Parameters used in the simulation

3-2) Defining the Fuel and Product for Each Component

In applying the SPECO approach, the fuel and product must be defined for each component. The fuel represents the resources required to generate the product and the product is what we desire from a component. Both the fuel and the product are expressed in terms of exergy.

3-3) Cost Balances

A cost balance states that the sum of cost rates associated with all exiting exergy streams equals the sum of cost rates of all entering exergy streams plus the cost rate associated with the capital investment and operating and maintenance costs (Z_k) . For each flow line in the system, a parameter called flow cost rate C(\$/s) is defined and the cost balance equation for a component that receives heat and produces power is written as :

$$\sum_{e} \dot{C}_{e,k} + \dot{C}_{w,k} = \dot{C}_{q,k} + \sum_{i} \dot{C}_{i,k} + \dot{Z}_{k}$$
$$\dot{C}_{j} = c_{j} \dot{E}_{j}$$

For calculating the cost of exergy destruction in the components of the system, first we solve the cost balance equations for each one. Generally, if there are N exergy streams exiting the component, we have N unknowns and only one equation, the cost balance. Therefore, we need to formulate N-1 auxiliary equations. This is performed with the aid of the F and P principles in the SPECO approach.

Developing cost balance equation for each component of the system and auxiliary equations (according to F and P rules) leads to a linear system of equations. By solving this, the costs of unknown streams are obtained.

Exergoeconomic assessments of systems can be performed using exergoeconomic parameters:

$$c_{F,k} = \frac{\dot{C}_{F,k}}{\dot{E}_{F,k}}$$

$$\dot{C}_{D,k} = c_{F,k} \dot{E}_{D,k}$$

 $c_{P,k} = \frac{\dot{C}_{P,k}}{\dot{E}_{P,k}}$

$$f_k = \frac{\dot{Z}_k}{\dot{Z}_k + \dot{C}_{D,k} + \dot{C}_{L,k}}$$

4) Results and Discussion

| , , | Table 2. Cost of streams in the combined cycles | | | | | | | | |
|---------------------|---|----------------|----------------|----------------|----------------|----------------|----------------|--|--|
| | | GT-MH | R/SORC | GT-MHR/HORC | | GT-MHR/RORC | | | |
| 4-1) Exergoeconomic | State no. | Ċ (\$/s) | c (\$/GJ) | Ċ (\$/s) | c (\$/GJ) | Ċ (\$/s) | c (\$/GJ) | | |
| | 1 | 17.17 | 11.83 | 17.15 | 11.83 | 17.20 | 11.83 | | |
| Analysis | 2 | 10.55 | 11.83 | 10.53 | 11.83 | 10.59 | 11.83 | | |
| 1 mary 515 | 3 | 7.428 | 11.83 | 7.419 | 11.83 | 7.444 | 11.83 | | |
| | 4 | 7.016 | 11.83 | 7.015 | 11.83 | 7.046 | 11.83 | | |
| | 5 | 6.936 | 11.83 | 6.927 | 11.83 | 6.953 | 11.83 | | |
| | 6 | 8.565 | 12.15 | 8.558 | 12.15 | 8.582 | 12.15 | | |
| | 7 | 8.347 | 12.15 | 8.338 | 12.15 | 8.362 | 12.15 | | |
| | 8 | 10.05 | 12.39 | 10.04 | 12.39 | 10.06 | 12.39 | | |
| | 9 | 13.18 | 12.56 | 13.17 | 12.56 | 13.22 | 12.56 | | |
| | 10 | 0.010 | 32.46 | 0.0009 | 18.5 | 0.0008 | 18.05 | | |
| | 11 | 0.434 | 18.36 | 0.010 | 32.61 | 0.001 | 24.10 | | |
| | 12 | 0.045 | 18.36 | 0.021 | 36.05 | 0.007 | 24.22 | | |
| | 13 | 0.0009 | 18.36 | 0.438 | 18.50 | 0.016 | 28.98 | | |
| | 14 | 0 | 0 | 0.046 | 18.50 | 0.427 | 18.05 | | |
| | 15 | 0.085 | 72.86 | 0.039 | 18.50 | 0.006 | 18.05 | | |
| | 16 | 0 | 0 | 0 | 0 | 0.042 | 18.05 | | |
| | 17 | 0.222 | 59.80 | 0.093 | 66.88 | 0 | 0 | | |
| | 18 | 0 | 0 | 0 | 0 | 0.098 | 64.10 | | |
| | 19 | 0.050 | 47.9 | 0.224 | 59.69 | 0 | 0 | | |
| | 20 | - | - | 0 | 0 | 0.224 | 59.56 | | |
| | 21 | - | - | 0.044 | 45.52 | 0 | 0 | | |
| | 22 | - | - | - | - / | 0.046 | 50.73 | | |
| | Nuclear fuel | 2.424 | 4.040 | 2.422 | 4.036 | 2.422 | 4.036 | | |
| | Ψ _T Ψ _{C,HP} | 6.843 1.695 | 12.56 12.56 | 6.843 1.695 | 12.55 12.55 | 6.837 1.692 | 12.56 12.56 | | |
| | Ŵ _{C,LP} | 1.622 | 12.56 | 1.624 | 12.55 | 1.622 | 12.56 | | |
| | Ŵ _{T,ORC} | 0.458 | 26.68 | 0.461 | 26.89 | 0.449 | 26.21 | | |
| | W _{P,ORC} | 0.0085 | 26.68 | 0.0085 | 26.89 | 0.0006 | 26.21 | | |
| | Ŵ _{P2,ORC} | - | - | | - | 0.008 | 26.21 | | |

15

| | GT-MHR/SORC | | | | GT-MHR/HORC | | | | GT-MHR/RORC | | | |
|---------------|----------------|-------|----------------|-------|----------------|-------|----------------|-------|----------------|-------|----------------|-------|
| Component | Ė _D | 3 | Ċ _D | f | Ė _D | 3 | Ċ _D | f | Ė _D | 3 | Ċ _D | f |
| | (kW) | (%) | (\$/s) | (%) | (kW) | (%) | (\$/s) | (%) | (kW) | (%) | (\$/s) | (%) |
| Reactor core | 198088 | 87.99 | 1.874 | 45.51 | 198122 | 87.98 | 1.874 | 45.52 | 197980 | 88.02 | 1.874 | 45.51 |
| Turbine | 14868 | 97.34 | 0.176 | 55.40 | 14878 | 97.34 | 0.176 | 55.37 | 14837 | 97.35 | 0.176 | 55.54 |
| Recuperator | 25397 | 90.37 | 0.301 | 4.262 | 25315 | 90.38 | 0.299 | 4.275 | 25605 | 90.36 | 0.303 | 4.238 |
| Evaporator | 11436 | 67.10 | 0.153 | 8.339 | 11035 | 67.64 | 0.131 | 9.154 | 10591 | 68.57 | 0.125 | 8.997 |
| Precooler | 5599 | 17.22 | 0.066 | 6.760 | 6054 | 18.65 | 0.072 | 6.281 | 6324 | 19.41 | 0.075 | 6.048 |
| LP compressor | 10536 | 91.84 | 0.132 | 5.180 | 10541 | 91.85 | 0.132 | 5.181 | 10520 | 91.86 | 0.132 | 5.186 |
| Intercooler | 14226 | 20.68 | 0.173 | 2.180 | 14368 | 20.71 | 0.175 | 2.158 | 14354 | 20.76 | 0.174 | 2.166 |
| HP compressor | 10830 | 91.98 | 0.136 | 5.119 | 10835 | 91.98 | 0.136 | 5.120 | 10815 | 91.98 | 0.136 | 5.125 |
| ORC Turbine | 4014 | 81.05 | 0.074 | 48.56 | 4013 | 81.03 | 0.074 | 48.37 | 6221 | 81.41 | 0.112 | 38.07 |
| Condenser | 1369 | 43.29 | 0.025 | 18.59 | 1081 | 46.91 | 0.020 | 22.54 | 1352 | 40.25 | 0.024 | 17.98 |
| Pump | 320 | 85.43 | 0.009 | 10.36 | 45.85 | 85.43 | 0.001 | 44.19 | 3.084 | 85.46 | 0 | 64.02 |
| Pump 2 | - | - | - | - | - | - | - | - | 43.87 | 85.88 | 0.001 | 45.69 |
| IHE | - | - | - | - | 135 | 66.15 | 0.002 | 56.32 | - | - | - | - |
| OFOF | - | - | - | - | - | - | - | - | 78 | 78.73 | 0.002 | _ |
| Overall | 296683 | 49.61 | 3.101 | 38.1 | 296425 | 49.58 | 3.092 | 38.22 | 298724 | 49.56 | 3.134 | 37.85 |

Table 3. Important exergy and exergoeconomic parameters of the combined cycles

16

The results show that the reactor core has the highest value of exergy destruction cost rate among the other components in all three combined cycles. The GT-MHR/RORC has the highest value of the exergy destruction cost rate and the lowest value of the unit cost of electricity produced by the ORC turbine. These results are reversed for GT-MHR/HORC.

4-2) Parametric Study



Effects of turbine inlet temperature (T_I) on the (a) unit cost of electricity produced by the ORC turbine $c_{W,T,ORC}$ and (b) total exergy destruction cost rate $\dot{C}_{D,total}$



Effects of compressor pressure ratio (PR_C) on the (a) unit cost of electricity produced by the ORC turbine $c_{W,T,ORC}$ and (b) total exergy destruction cost rate $\dot{C}_{D,total}$



Effects of evaporator temperature (T_E) on the (a) unit cost of electricity produced by the ORC turbine $c_{W,T,ORC}$ and (b) total exergy destruction cost rate $\dot{C}_{D,total}$

Parametric study shows that increasing turbine inlet temperature increases the unit cost of electricity produced by the ORC turbine and decreases the exergy destruction cost rate, however, these exergoeconomic parameters have a minimum value with respect to compressor pressure ratio and evaporator temperature in all three combined cycles.

5) Conclusions

A comparative exergoeconomic analysis of waste heat recovery from a Gas Turbine-Modular Helium Reactor (GT-MHR) using different arrangements of Organic Rankine Cycles (ORCs) for electrical power production is successfully performed. For this purpose, energy and exergy analyses of combined GT-MHR/ORC cycles are performed. Then, cost balances and auxiliary equations are applied to subsystems and exergoeconomic parameters are calculated for the components and entire combined cycles. Finally a parametric study is performed to reveal the effects of selected parameters on the exergoeconomic performance of the combined cycles. The considered organic Rankine cycles for electrical power production are: Simple Organic Rankine Cycle (SORC), ORC with internal heat exchanger (HORC) and Regenerative Organic Rankine Cycle (RORC).

The results show that the reactor core has the highest value of exergy destruction cost rate among the other components in all three combined cycles. The GT-MHR/RORC has the highest value of the exergy destruction cost rate and the lowest value of the unit cost of electricity produced by the ORC turbine. These results are reversed for GT-MHR/HORC. Also parametric study shows that increasing turbine inlet temperature increases the unit cost of electricity produced by the ORC turbine and decreases the exergy destruction cost rate, however, these exergoeconomic parameters have a minimum value with respect to compressor pressure ratio and evaporator temperature in all three combined cycles.

The results of the present work can be used as a basis for the exergoeconomic optimization of the considered combined cycles.

Thanks

for

your attention