

Proceeding

A Novel Geometry for Shear Test Using Axial Tensile Setup [†]

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Abstract: This paper studies a novel geometry for the in-plane shear test performed with an axial electromechanical testing machine. In order to investigate the influence of the triaxiality rate on the mechanical behavior, different tests will be performed on the studied material: simple tensile tests, large tensile tests and shear tests. For the whole campaign, a common equipment should be employed to minimize the impact of the testing device. As a consequence, for the shear tests, the geometry of the specimen must be carefully designed in order to adapt the force value and make it comparable to the one obtained for the tensile tests. Like most of the existing shear-included tensile test specimens, the axial loading is converted to shear loading at a particular region through the effect of geometry. A symmetric shape is generally preferred, since it can restrict the in-plane rotation of the shear section, keep shear increasing in a more monotonic path and double the force level thanks to the two shear zones. Due to the specific experimental conditions, such as dimensions of the furnace and the clamping system, the position of the extensometer or the restriction of sheet thickness (related to the further studies of size effect at mesoscale and hot temperature), several geometries were brought up and evaluated in an iterative procedure via finite element simulations. Both the numerical and experimental results reveal that the final geometry ensures some advantages. For instance, a relatively low triaxiality in the shear zone, limited in-plane rotation and no necking are observed. Moreover, it also prevents any out-of-plane displacement of the specimen which seems to be highly sensitive to the geometry, and presents a very limited influence of the material and the thickness.

Keywords: In-plane shear; Thin specimen; Finite element simulation

1. Introduction

In the present study, a novel specimen geometry for the in-plane shear test is proposed. By benefiting from this exotic geometry, the shear test can be carried out with the common experimental equipment as the one used for the tensile tests and the plane strain tests. The interest of using a single press is to avert the uncertain impacts from the equipment for different loading paths

during the experimental campaign. The main objective to develop such in-plane shear test is to prepare the further investigation of the downscaling effects on the mechanical behavior at different triaxiality rates and various temperatures [1-4].

In order to comply with equipment constraints, the geometry of the shear test sample is required to be redesigned in order to adapt the force level to make it comparable to the one obtained for the tensile tests. Like most of the proposed geometries in the literature [5-9], the axial loading is converted to shear the material at a specific region thanks to the effects of the geometry. An iterative procedure is numerically performed by finite element simulations to determinate the best geometry yielding to a homogenous shear zone whose finality is assessed thanks to the triaxiality ratio T (pure shear corresponds to $T = 0$), free from the spurious deformation modes in the sample. The results are further validated by experiments with different materials and thicknesses of specimens.

In section 2, the numerical modeling is presented, and then the corresponding results are exhibited in the section 3. The experimental validation is shown in section 4. In the end, a succinct conclusion is addressed in section 5.

2. Numerical modeling

Considering the potential in-plane rotation of the shear zone, a symmetric geometry is preferred as shown in Figure 1 (a). By reason of the constraints of experimental conditions, such as the dimension of the furnace and the clamping device, the position of the extensometer's arm etc., the global dimensions (in mm) are defined and shown in Figure 1 (a). As the final goal is to investigate the scale effect under different strain paths and temperatures, this experimental campaign have to deal with sheet thickness of 0.5 mm to follow the previous researches [2-4; 10]. The shaded area denotes the grip zone and will not be considered in the simulation. The parameters l and h refer to the width between the two shear zones and the height of the latter respectively. The mesh and the boundary conditions for the middle part with the height of 80 mm are illustrated in Figure 1 (b).

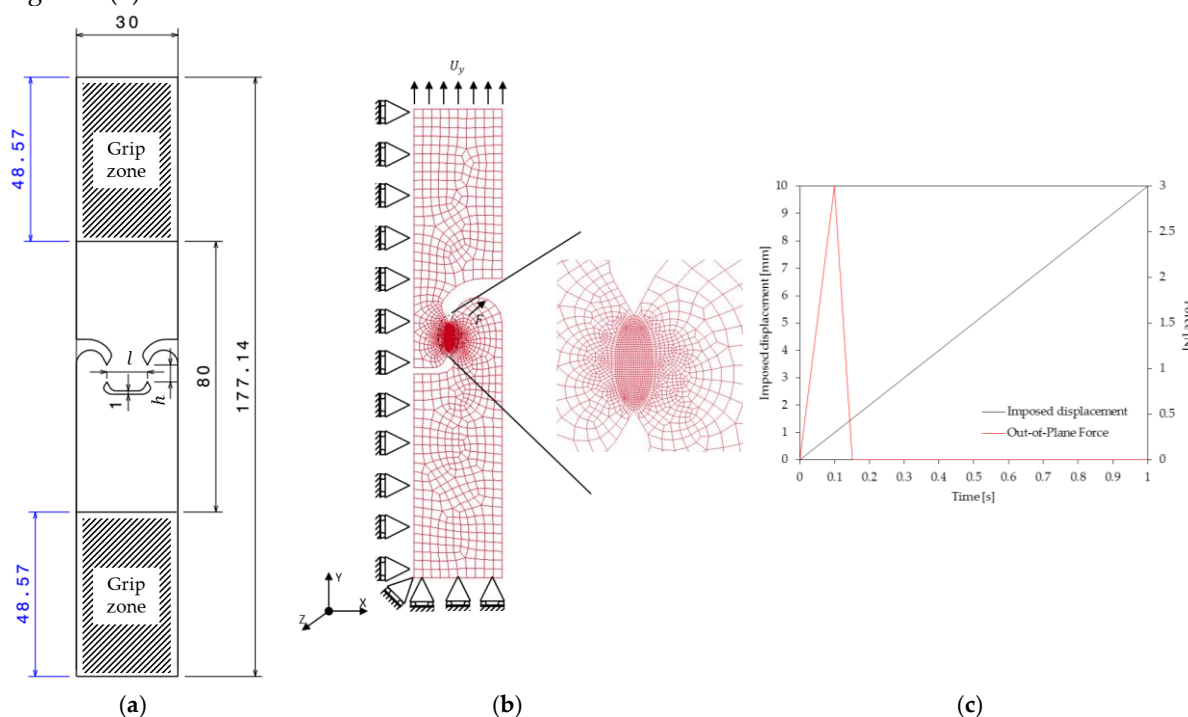


Figure 1. (a) Sketch of the in-plane shear test specimen with dimensions in mm; (b) delineation of the boundary conditions and mesh employed in the numerical simulations; the zoom illustrates the refined mesh in the shear zone; (c) evolution of imposed displacement and out-of-plane force.

The symmetry about Y-Z plane (Figure 1 (b)) is taken into account; hence only a half of the sample is modeled. The imposed displacement at the top surface acts as the driving force to deform the specimen. An extra out-of-plane force is applied near the shear zone (see Figure 1 (b)) aiming to

break the perfect symmetry of the model along the thickness direction to trigger the possible out-of-plane deformation. As shown in Figure 1 (c), the out-of-plane force is merely applied at the beginning of the simulation and its magnitude remains relatively small, the peak value is 3 N. Once the deformed geometry is eventually unsymmetric, the axial load continues to deform the specimen. The imposed displacement U_y evolves linearly from 0 to 10 mm throughout the simulation.

An isotropic elastoplastic constitutive law (the hardening behavior is expressed by equation (1)) is utilized and the relevant parameters are listed in Table 1.

$$\sigma = c\epsilon^n \tag{1}$$

Table 1. Material parameters utilized in the simulation.

Material parameters	Symbols	Unit	Magnitude
Young’s modulus	E	MPa	210000
Poisson’s ratio	ν	–	0.31
Hardening constant	c	MPa	934.25
Exponent hardening	n	–	0.57157

3. Numerical results

3.1. Influence of l and h parameters

Figure 2 depicts the deformed configurations for different l and h values. The parameter L denotes the initial width of the specimen which is 30 mm as shown in Figure 1 (a). The fringe levels refer to the displacement field along the loading direction. The material inside the shear zone is supposed to be purely sheared when one side of the shear zone moves vertically and the other side remains immobile. It is additionally desired that such displacement appears without generating any necking zone in the sample. Hence, the geometries with l/L ratios of 0.25, 0.33 and 0.5 are already eliminated.

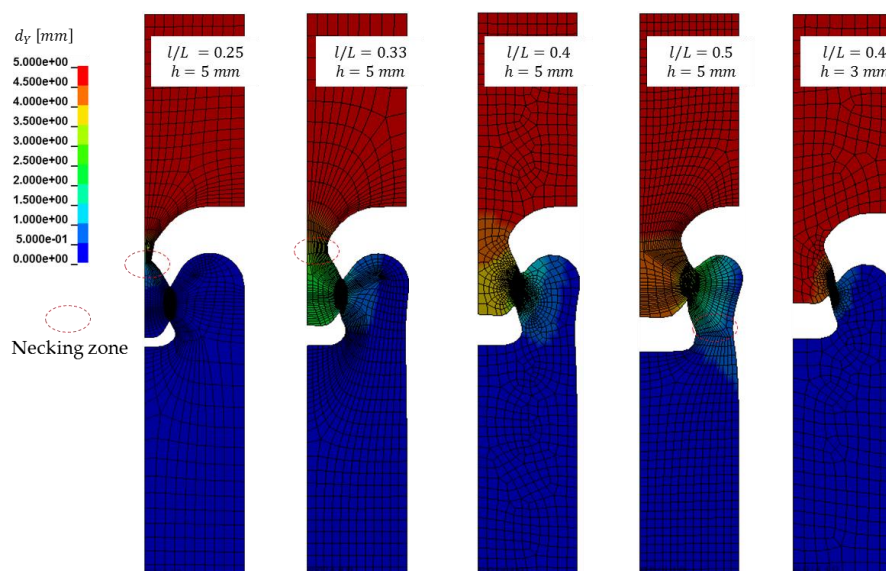


Figure 2. Contours of the vertical displacement for different dimensions of the shear zone, for $U_y = 5$ mm.

According to the results of Figure 3, an appropriate height of the shear zone seems to be more critical to obtain the targeted triaxiality ratio rather than to the out-of-plane deformation. Reducing the height of the shear zone could decrease a little the out-of-plane displacement but risks of resulting in an unstable triaxiality since the shear zone is quickly deformed and tends to rupture. The insert fringe level image in Figure 3 (b) shows an example of the triaxiality field for a given

imposed displacement, $U_Y = 9 \text{ mm}$. A homogenous distribution of the latter could be obtained with a height of the shear zone of 5 mm.

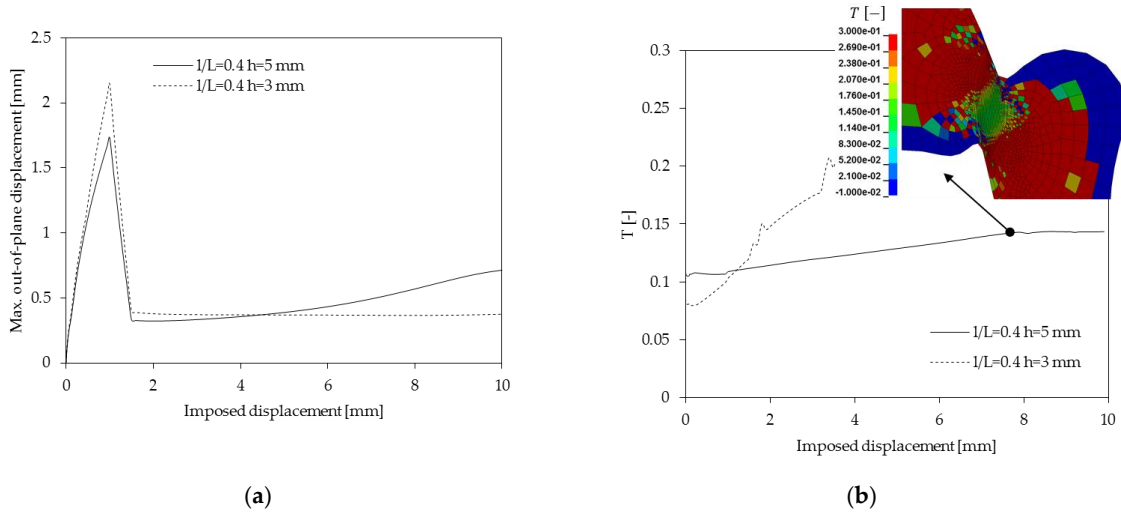


Figure 3. (a) Evolution of maximum out-of-plane displacement; (b) Evolution of triaxiality ratio, it is computed from the element situated in the center of the shear zone, the insert figure shows an example of the triaxiality field for the geometry with $h = 5 \text{ mm}$ (at $U_Y = 9 \text{ mm}$).

3.2. Influence of shift of shear zone

From section 3.1, only the geometry with a l/L ratio of 0.4 and a height of the shear zone h of 5 mm seems workable. Two variants of this geometry are proposed by shifting the shear zone of 0.5 mm and 1 mm respectively, as presented in Figure 4.

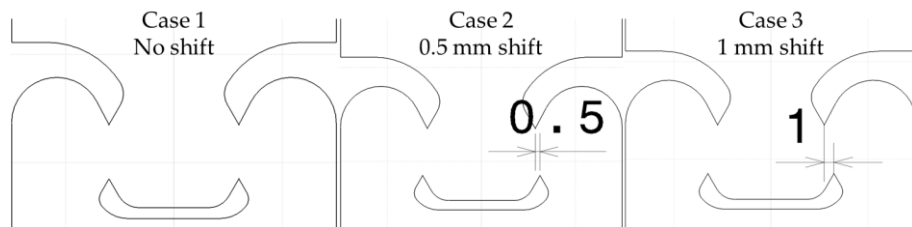


Figure 4. Sketch of non-shifted and shifted geometries. In each of the three cases, the l/L ratio and the height of the shear zone h are fixed as 0.4 and 5 mm, respectively.

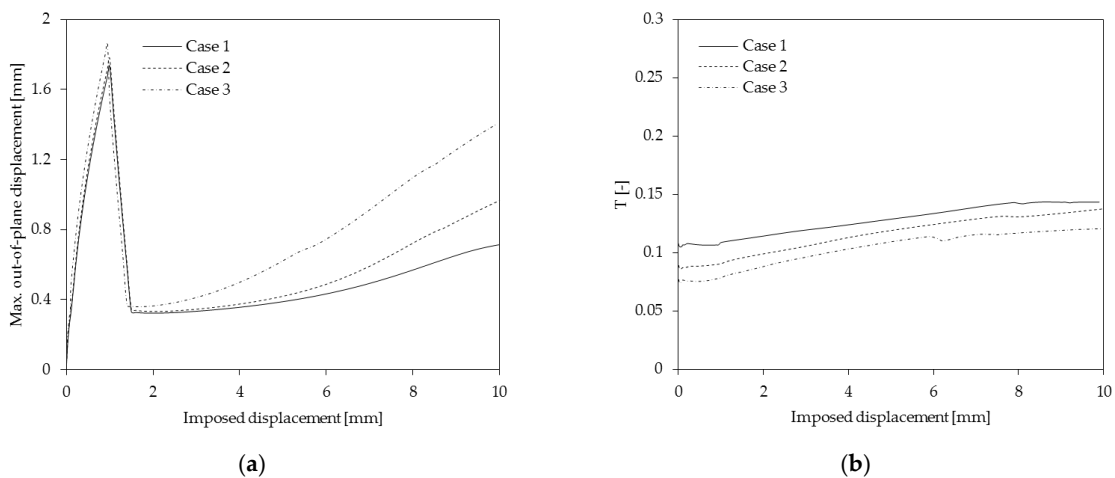


Figure 5. Evolution of (a) maximum out-of-plane displacement; (b) triaxiality.

Regarding the triaxiality ratio, it could be slightly reduced by shifting the shear zone as revealed in Figure 5 (b). However, the shift induces a larger shear section which, in turn, is likely to produce the out-of-plane deformation (Figure 5 (a)). Consequently, the in-plane shear could be jeopardized by a shifted shear zone, as confirmed experimentally hereunder.

4. Experimental results

The aforementioned geometries with non-shifted and shifted shear zones are tested by the experiments at room temperature with different materials and thicknesses of the specimen. As shown in Figure 6, aluminum and copper samples were tested. The different values of the shift of the shear zone are solely employed for the copper samples. The thicknesses of the aluminum and copper samples are 1 mm and 0.5 mm respectively. As revealed by the numerical studies, the shifted shear zones yield a visible out-of-plane deformation as demonstrated in Figure 6 (a). In contrast, the non-shifted case could ensure a quasi-purely sheared zone regardless of the material and the thickness of the samples. The evolution of the tensile force is drawn in Figure 7 as a function of the imposed displacement for the non-shifted shear zone (case 1).

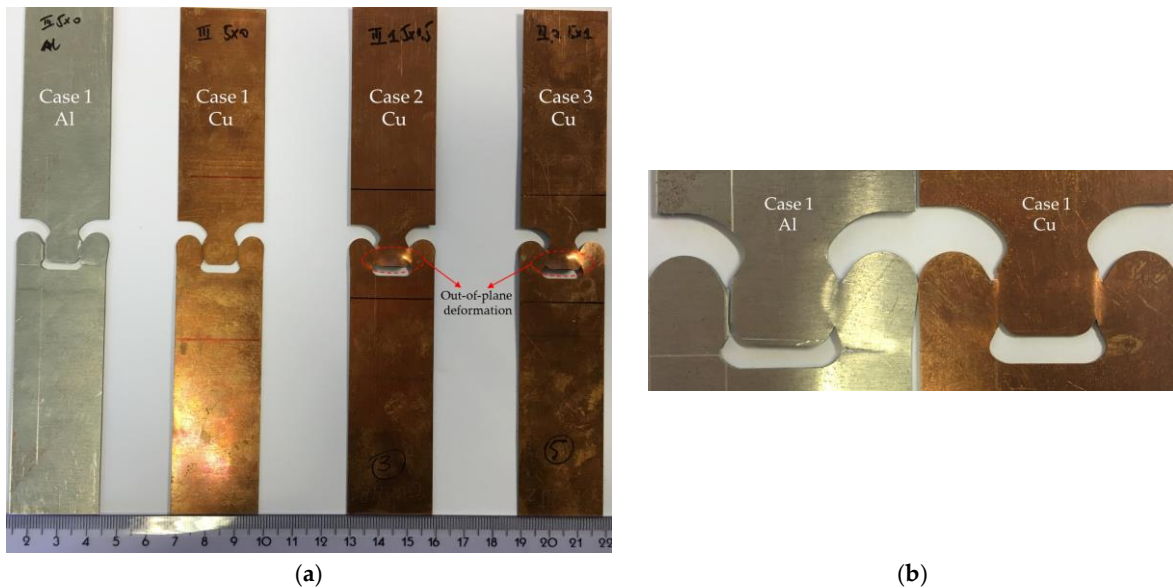


Figure 6. (a) Examples of deformed specimen for the non-shifted and shifted shear zone; (b) zoom of the shear zone for the case 1 (no shift).

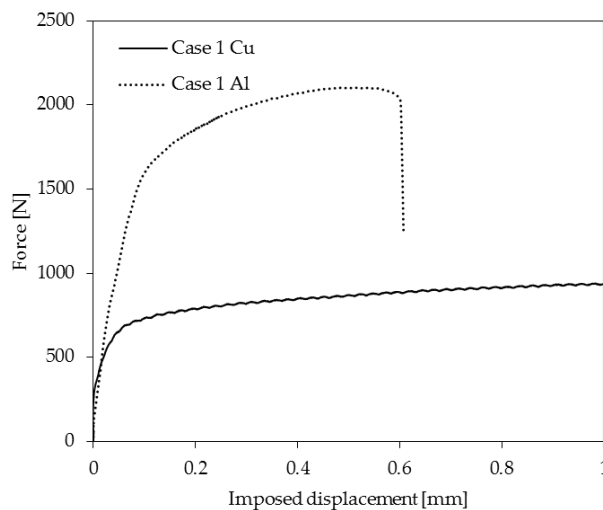


Figure 7. Force curve in function of displacement.

5. Conclusion

Through an iterative procedure, the final geometry proposed in this study provides a relatively low triaxiality and remains stable during the hardening stages, allowing the material in the designed region to be sheared in a monotonic path. In addition, the triaxiality field in the shear zone is fairly homogenous. Both the numerical and experimental results reveal that the studied geometry could prevent any out-of-plane displacement of the specimen which seems to be thickness and material independent but highly sensitive to the geometry. The in-plane rotation could also be drastically limited using a symmetric sample.

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Conflicts of Interest: The authors declare no conflict of interest.

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