

## Modeling of Fluid Damping in Resonant Micro-mirrors with Out-of-plane Comb-drive Actuation



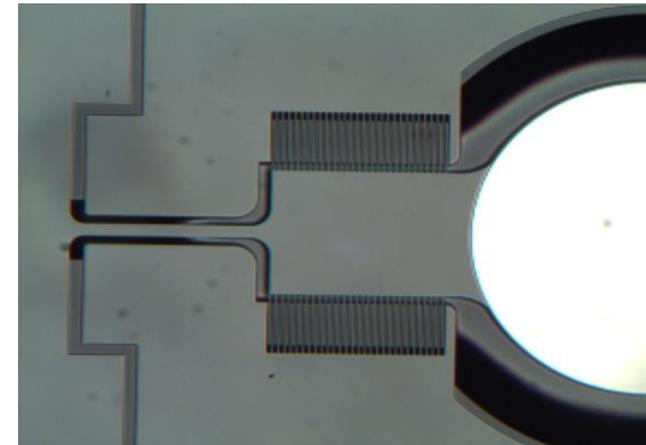
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# Micro-Electro-Mechanical Systems (MEMS)

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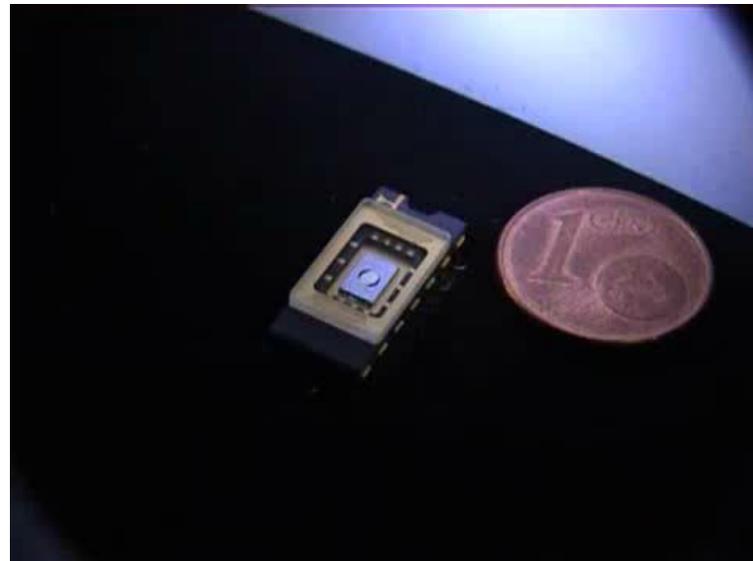
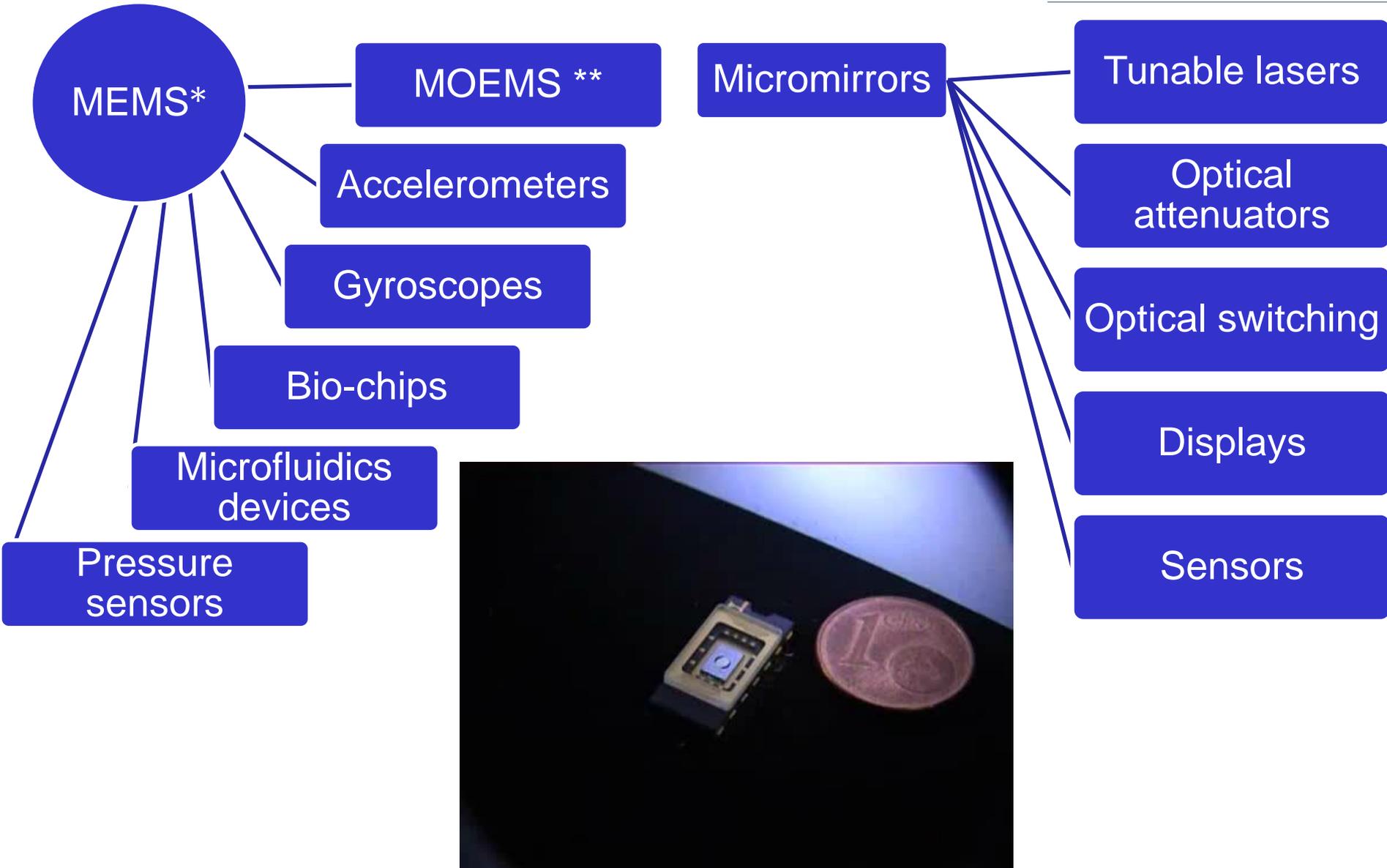


Image by Fraunhofer IPMS

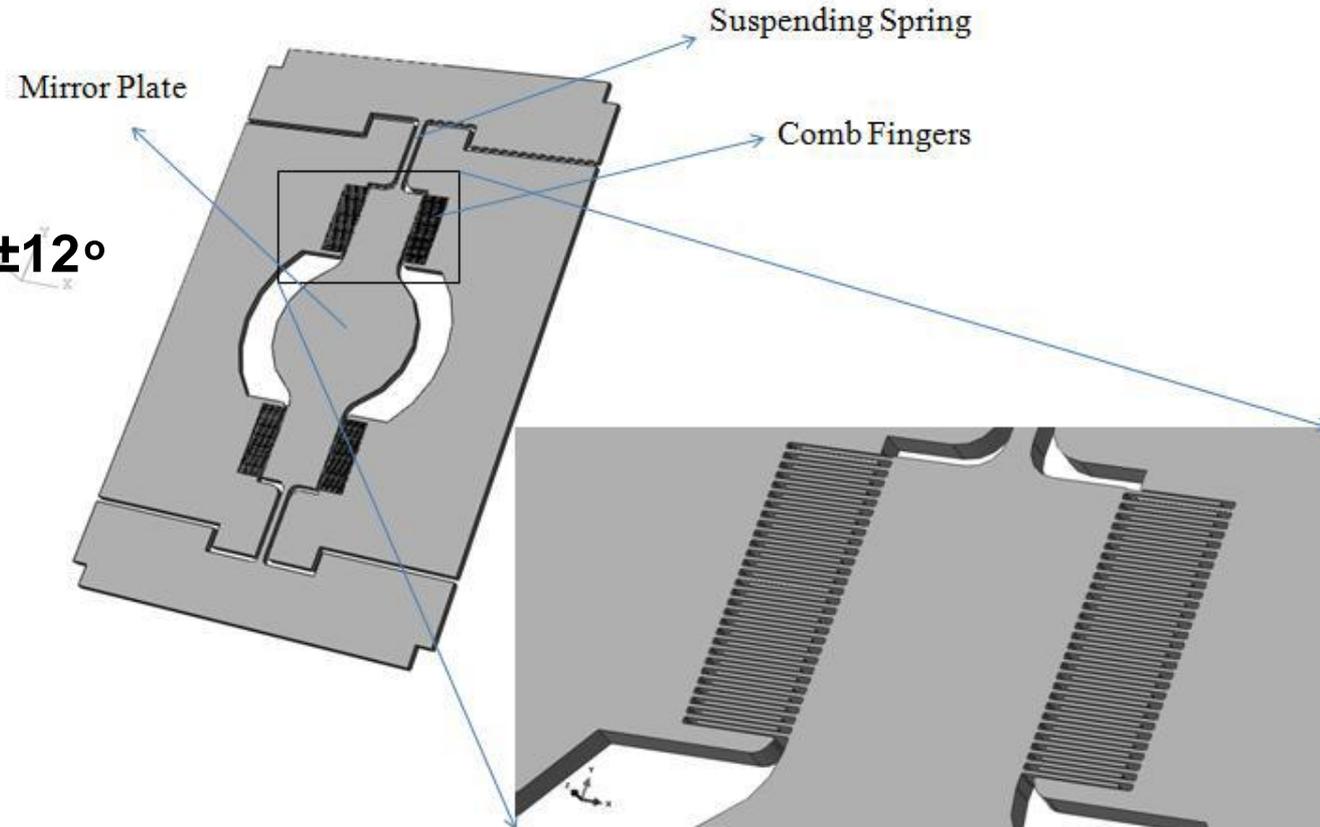
\* Micro-Electro-Mechanical Systems

\*\* Micro-Opto-Electro-Mechanical Systems

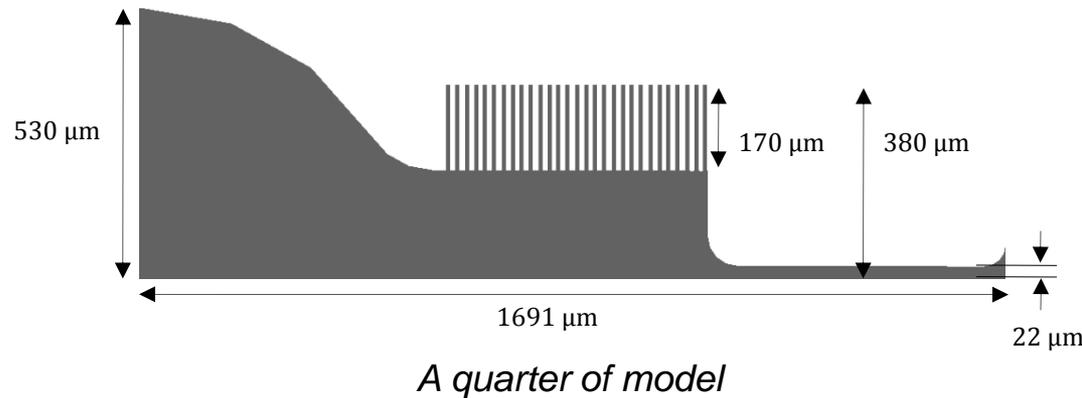
Ramin Mirzazadeh

- Air-packaged
- Working frequency 19 343 Hz
- Made by *STMicroelectronics*

- **Large tilting angle  $\pm 12^\circ$**

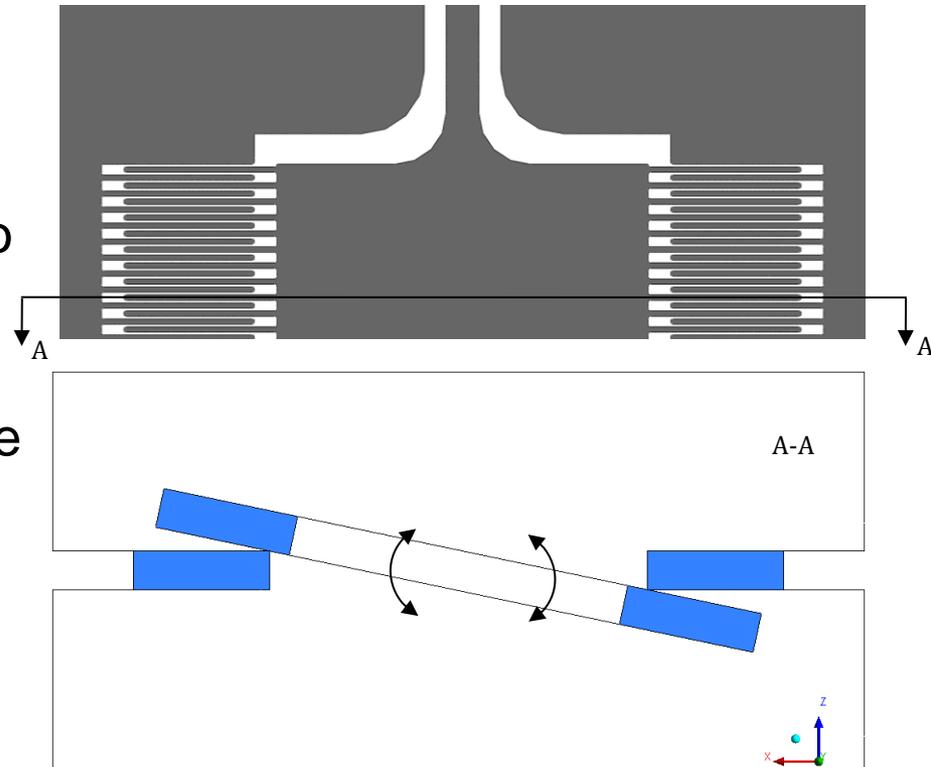


- Out-of-plane oscillation of combfingers
- Large angle oscillation
- What happens to the fluid when the structure has complex geometry and motion?
- 3D CFD

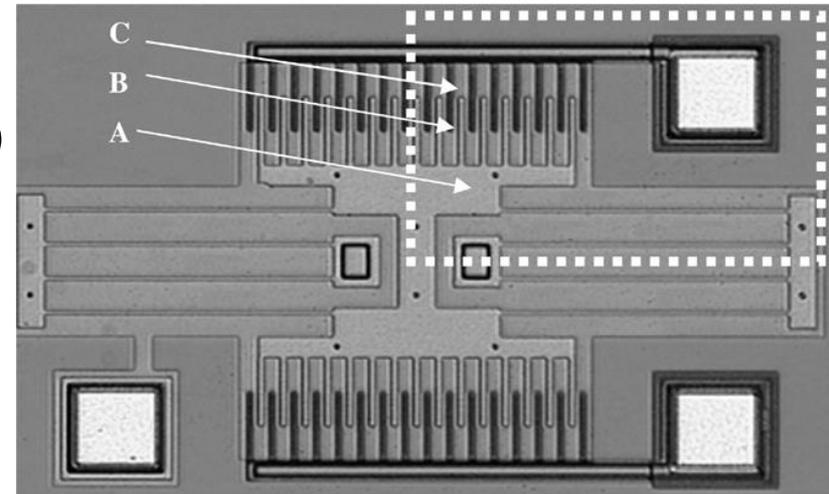


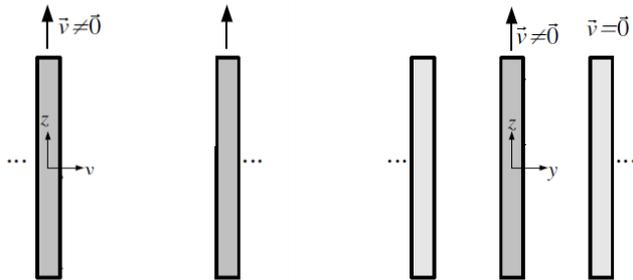
Parameter	Value
Mirror diameter	1060 μm
Spring length	579.5 μm
Spring width	44 μm
Finger length	170 μm
Finger width	6 μm
Finger span	760 μm
Finger gap	3 μm
Number of fingers	29
Thickness of layout	50 μm
Substrate depth	450 μm

- Damping behavior is important in microscales (surface forces)
  - Mass= $1.81 \cdot 10^{-7}$  kg
  - Moment of inertia= $9.384 \cdot 10^{-14}$  kg m<sup>2</sup>
- The energy dissipation is expected to be lower than the one for the small oscillation
- Experiments show no reduction in the energy dissipation

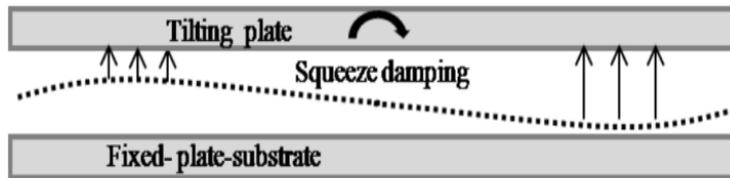
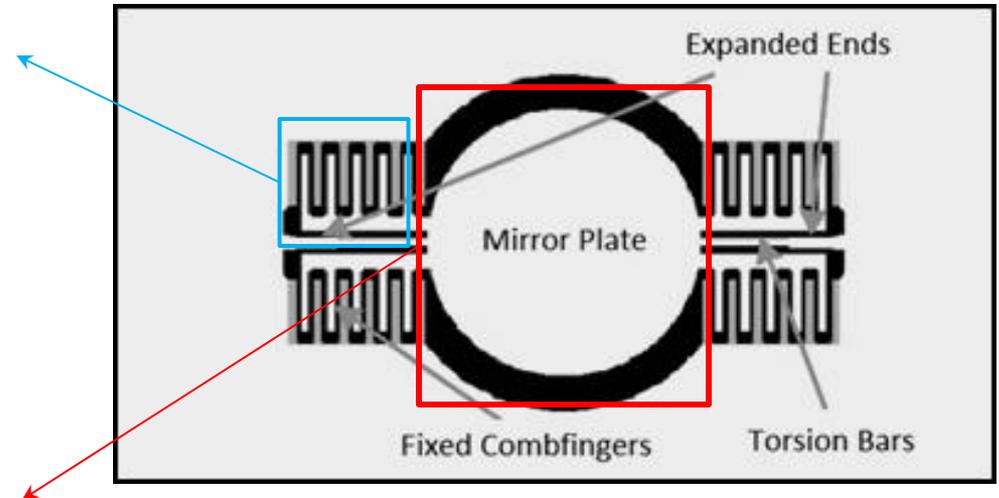


- **Howe, Pisano et al** (JMEMS, 1993)
  - 1D Stokes flow
- **Zhang and Tang** (IEEE on MEMS, 1994)
  - The combfinger effect in experiments
- **Ye, Werner, White et al** (JMEMS, 2003)
  - 3D Stokes solver (3D BEM), Stick BC
- **Sudipto and Aluru** (JMM, 2006)
  - Coupled model, 2D compressible N\_V
- **Frangi et al** (J. numerical methods Eng., 2006, Sensors and actuators A,2009)
  - Fast multipole boundary element, Slip BC
  - Boundary Integral Equation, Free molecule regime
- **Braghin et al** (NoD., 2008)
  - 3D Incompressible N\_V, constant speed





Shear damping



Drag damping  
Squeeze film damping

- Reynolds number  $\ll 1$ ,
- Mach number  $\ll 1$ ,
- Knudsen number  $\approx 0.01$ ,

Laminar flow  
Incompressible flow  
Continuum method

Mean free path of  
air molecules

$$Kn = \frac{\lambda}{L}$$

Characteristic length

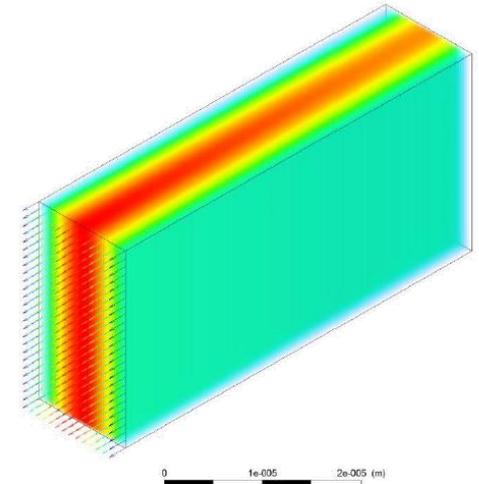
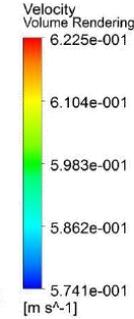
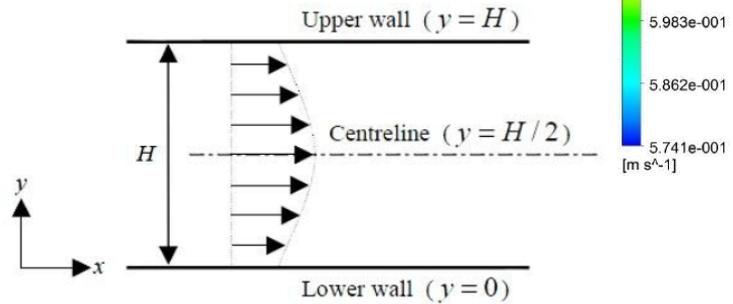


<b>Kn</b>	▶		<b>0.01</b>		<b>0.1</b>		<b>10</b>
<b>Regime</b>	▶		<b>Continuum</b>	<b>Slip</b>		<b>Transition</b>	<b>Free-molecule</b>

\*G.E. Karniadakis, A. Beskok, Micro Flows, Fundamentals and Simulation, Springer, New York, 2002

# Slip boundary condition- implementation in CFX

- ANSYS CFX
- User defined BC
- Error: 1.5%



TMAC

$$u_t = \frac{2 - \sigma}{\sigma} Kn 2H \left. \frac{du}{dy} \right|_{y=0}$$

Knudsen number

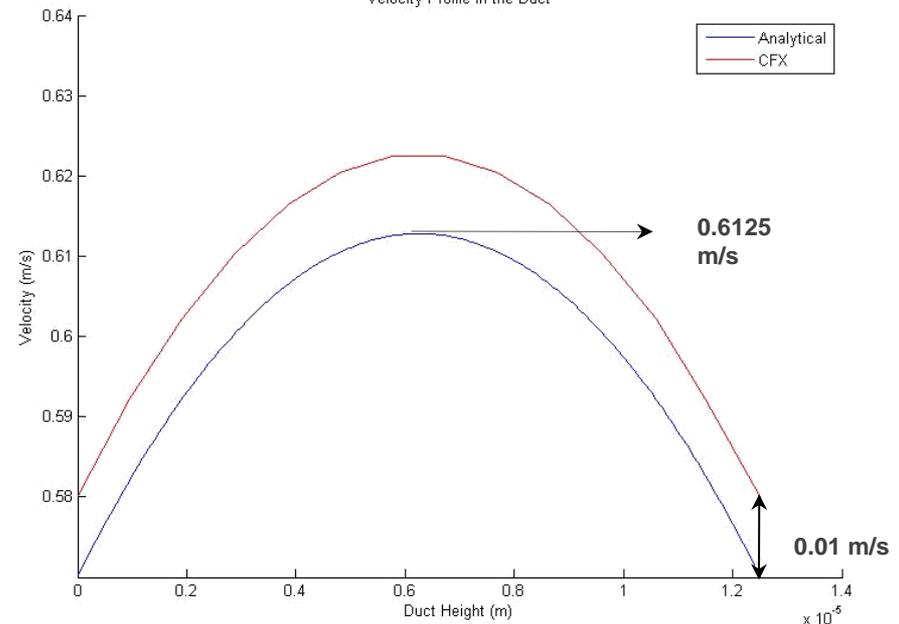
Boltzmann's constant

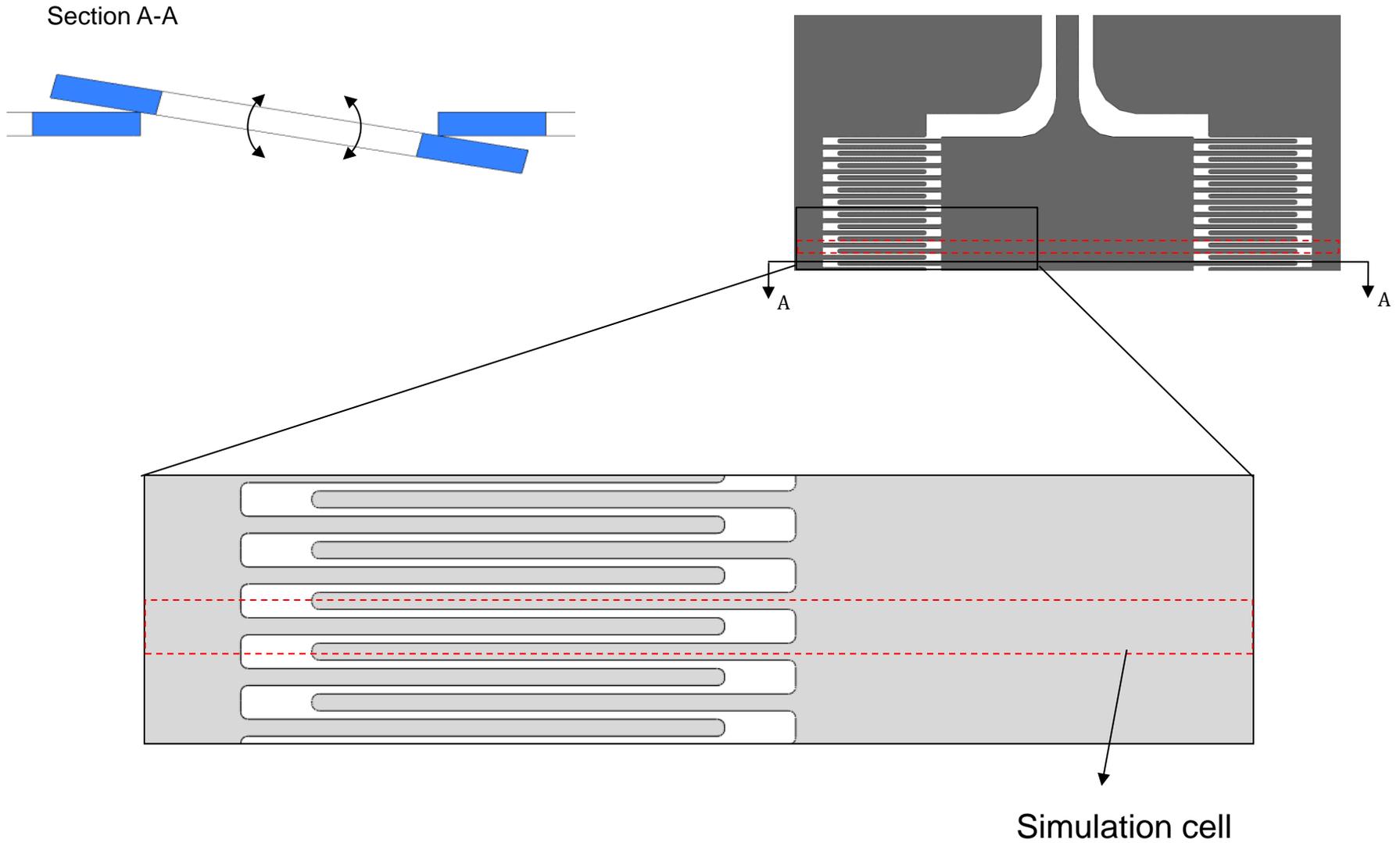
$$\lambda = \frac{\mathcal{R}T_K}{\sqrt{2}\pi p \mathcal{D}^2}$$

Averaged air molecule diameter

Mean free path of air molecules

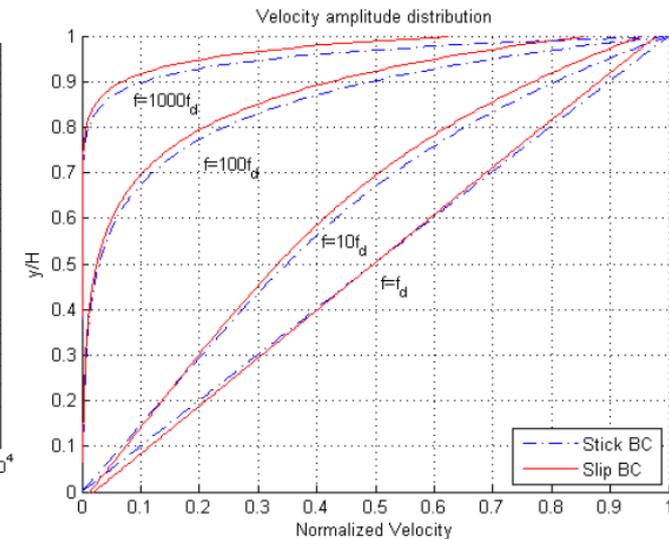
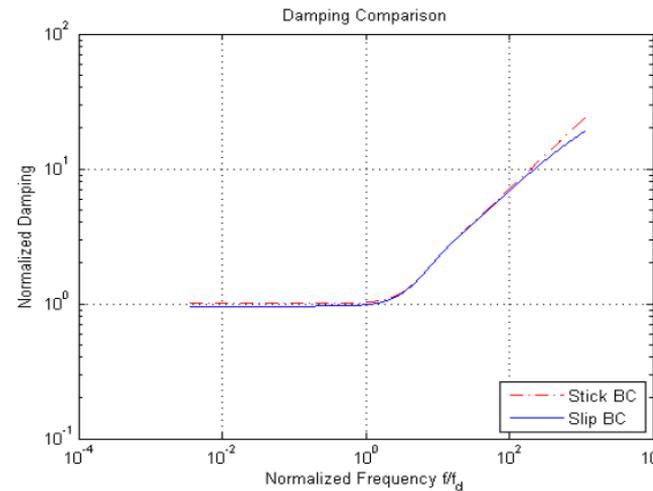
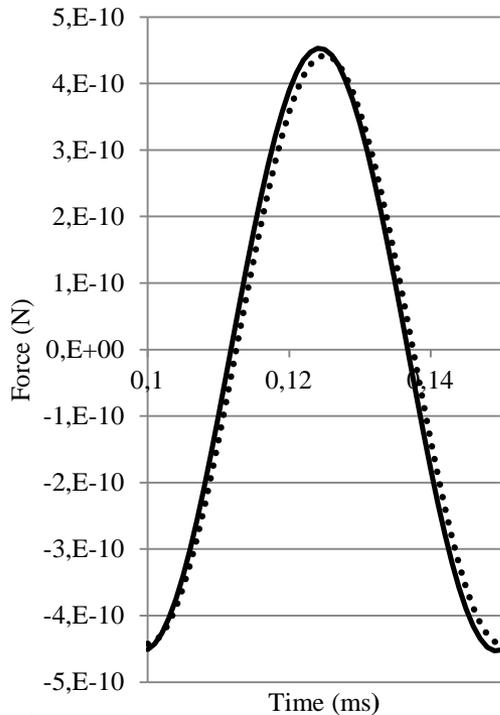
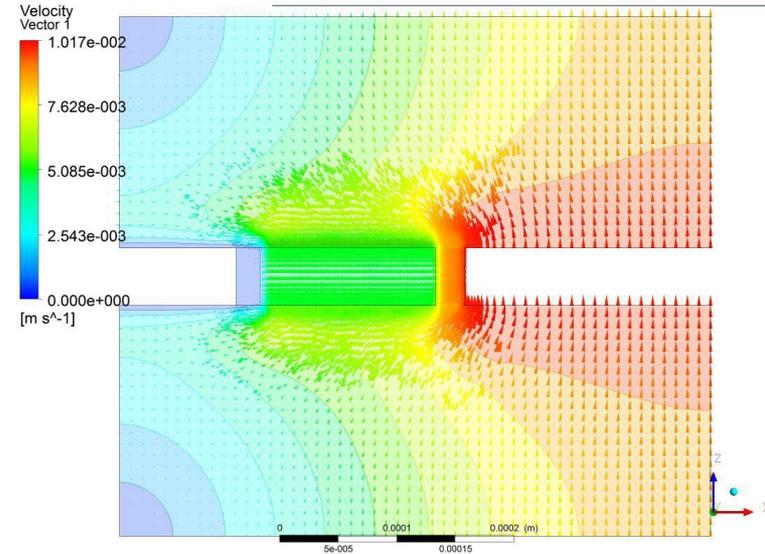
Velocity Profile in the Duct





# Validation for small displacements

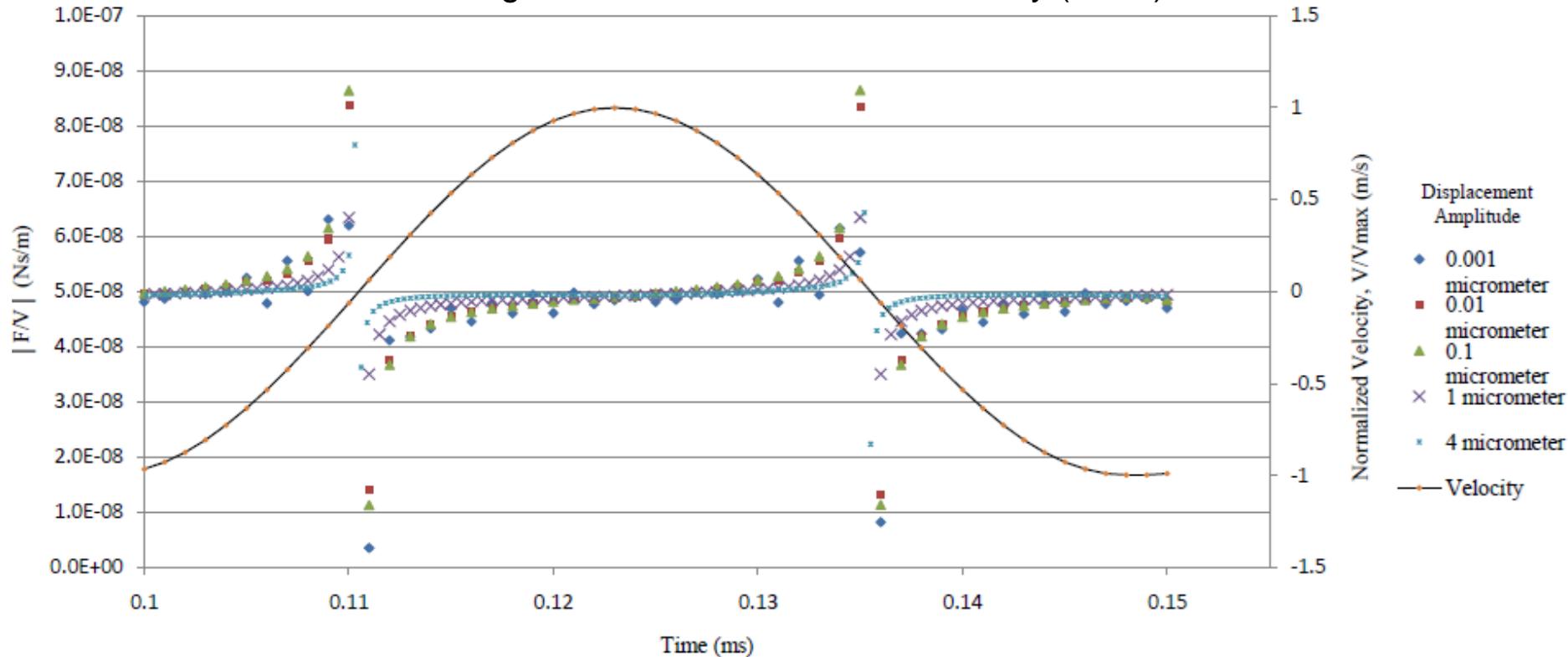
- Shear damping, small oscillation
- Couette flow model  $\{F_d\} \approx \{F_{Couette}\} = -\mu \frac{A_s}{g} \{u\}$
- 2% error



— Numerical solution  
 ..... Analytical solution

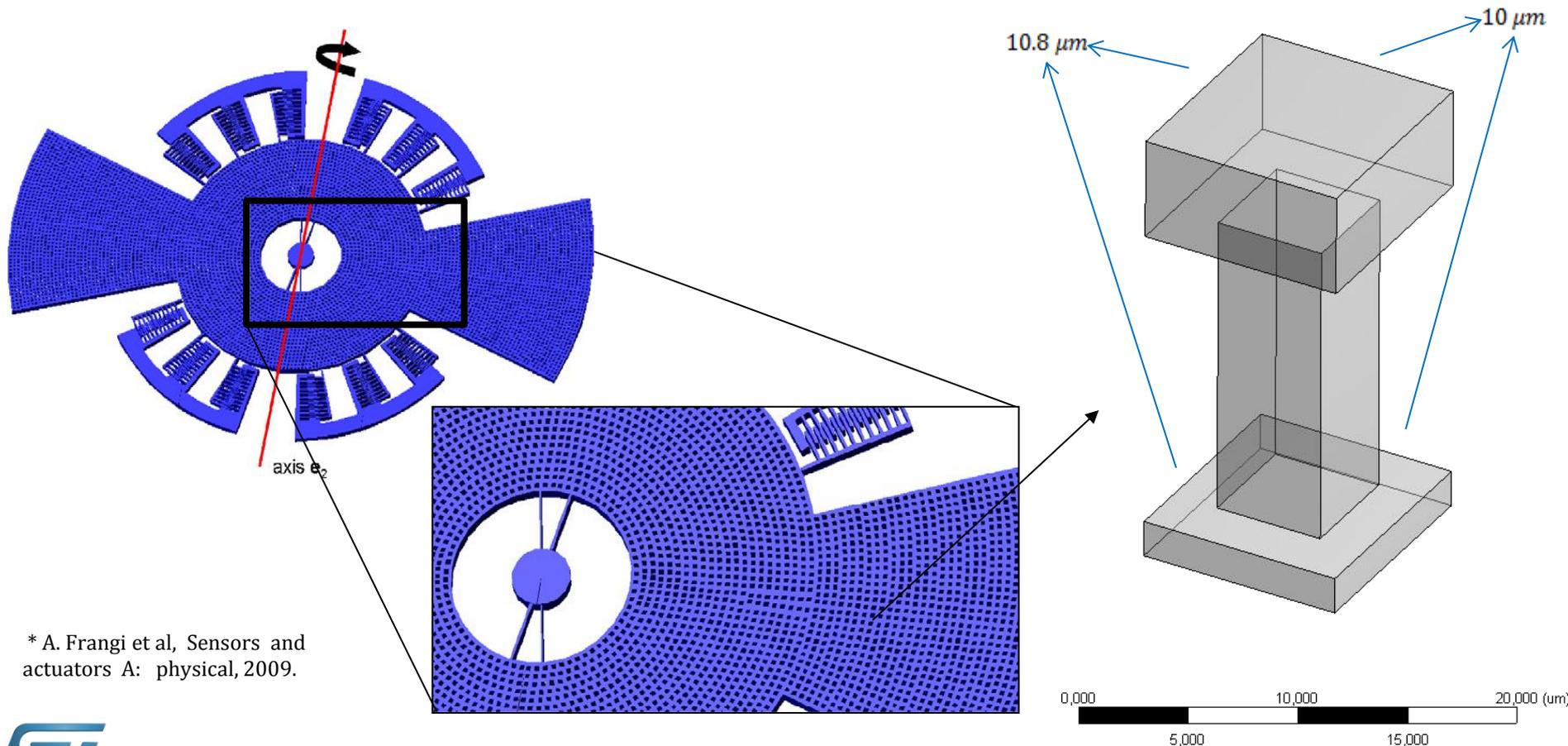
- Linear behavior
- Jumps when velocity is zero

Ratio between magnitude of shear force and velocity (Ns/m)



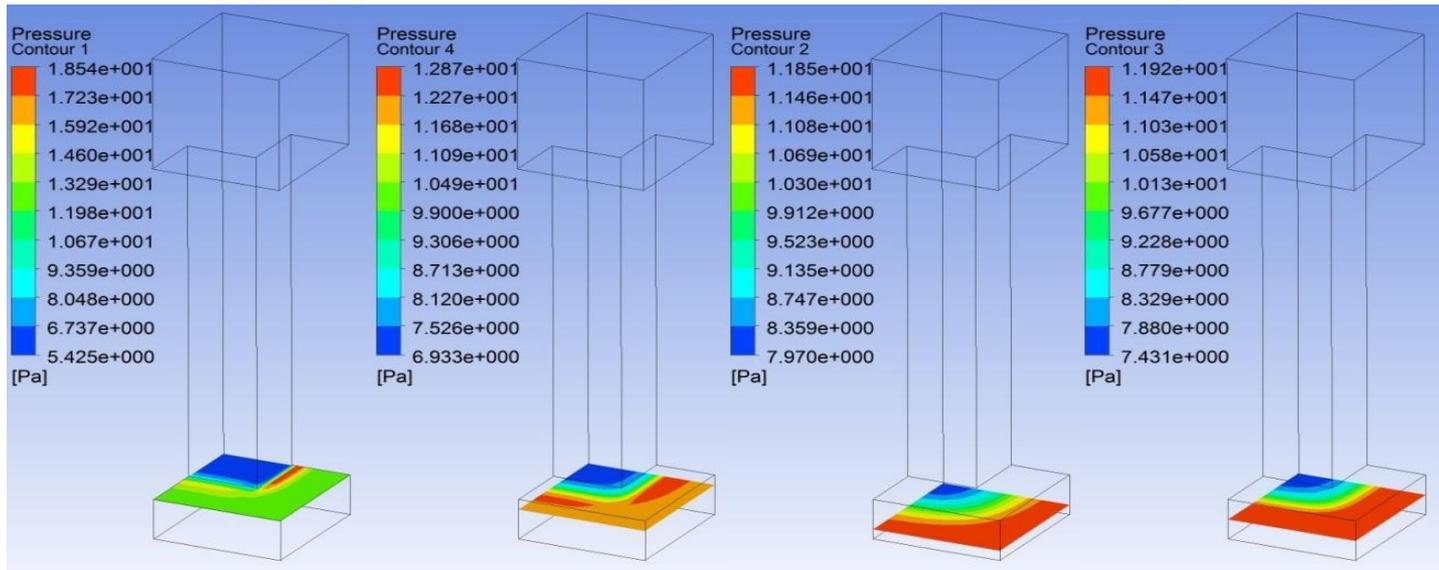
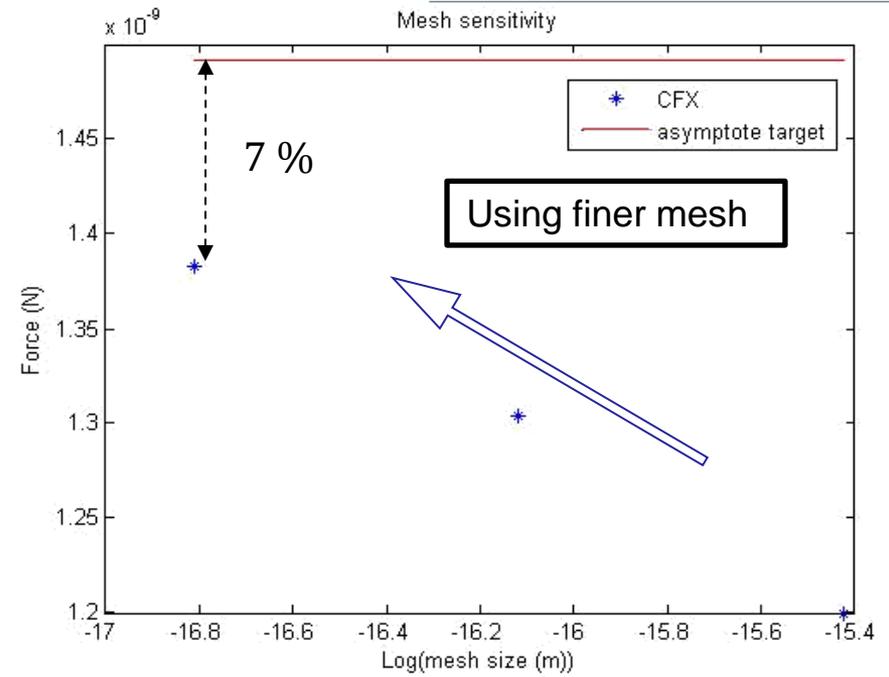
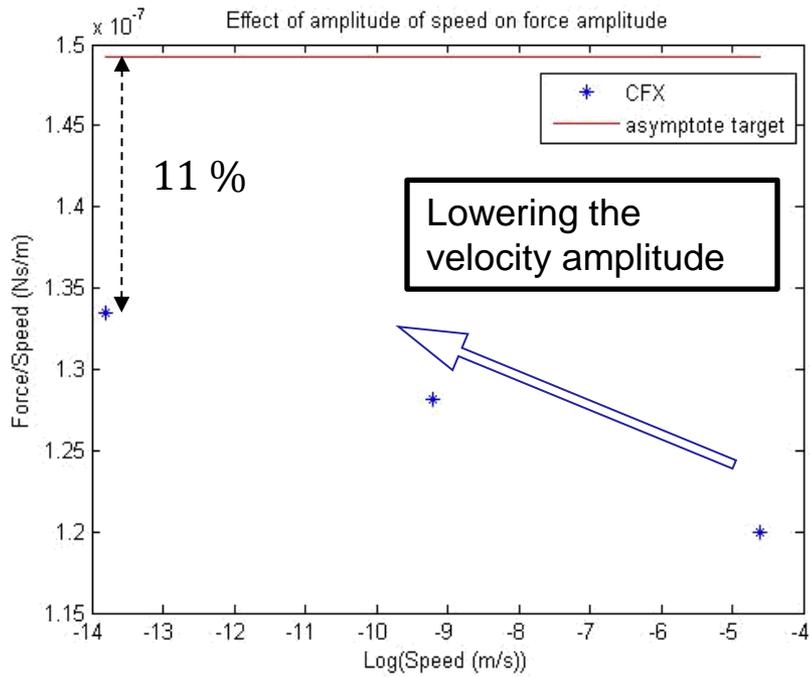
# Validation for squeeze film damping-benchmark\*

- Rotational resonator- perforation cell
- Resonance frequency: 4550 Hz
- Reported overall force on the unit cell for unit velocity:  $1.492 \cdot 10^{-7}$  N

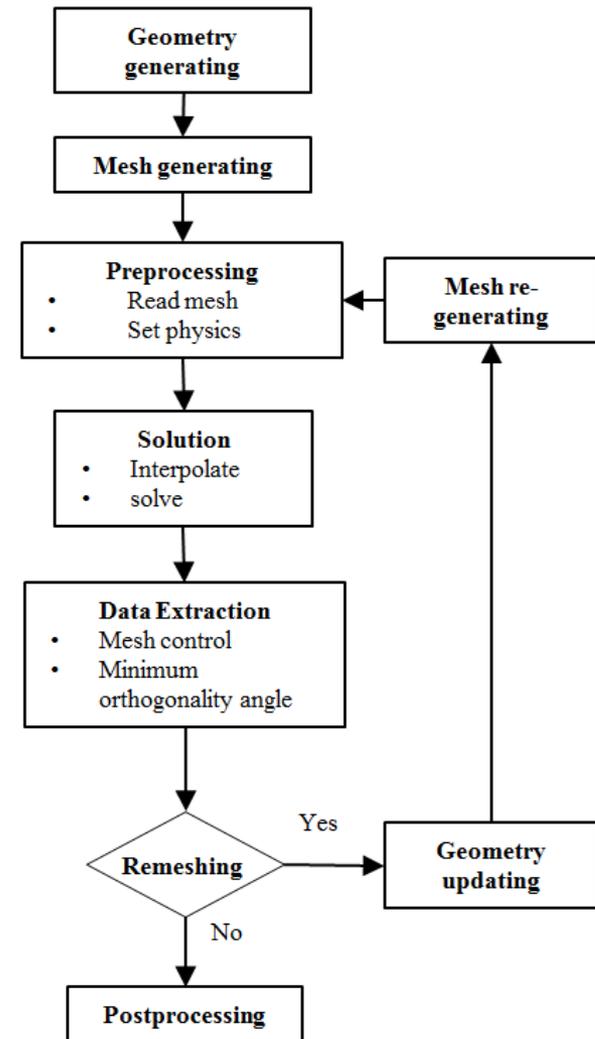


\* A. Frangi et al, Sensors and actuators A: physical, 2009.

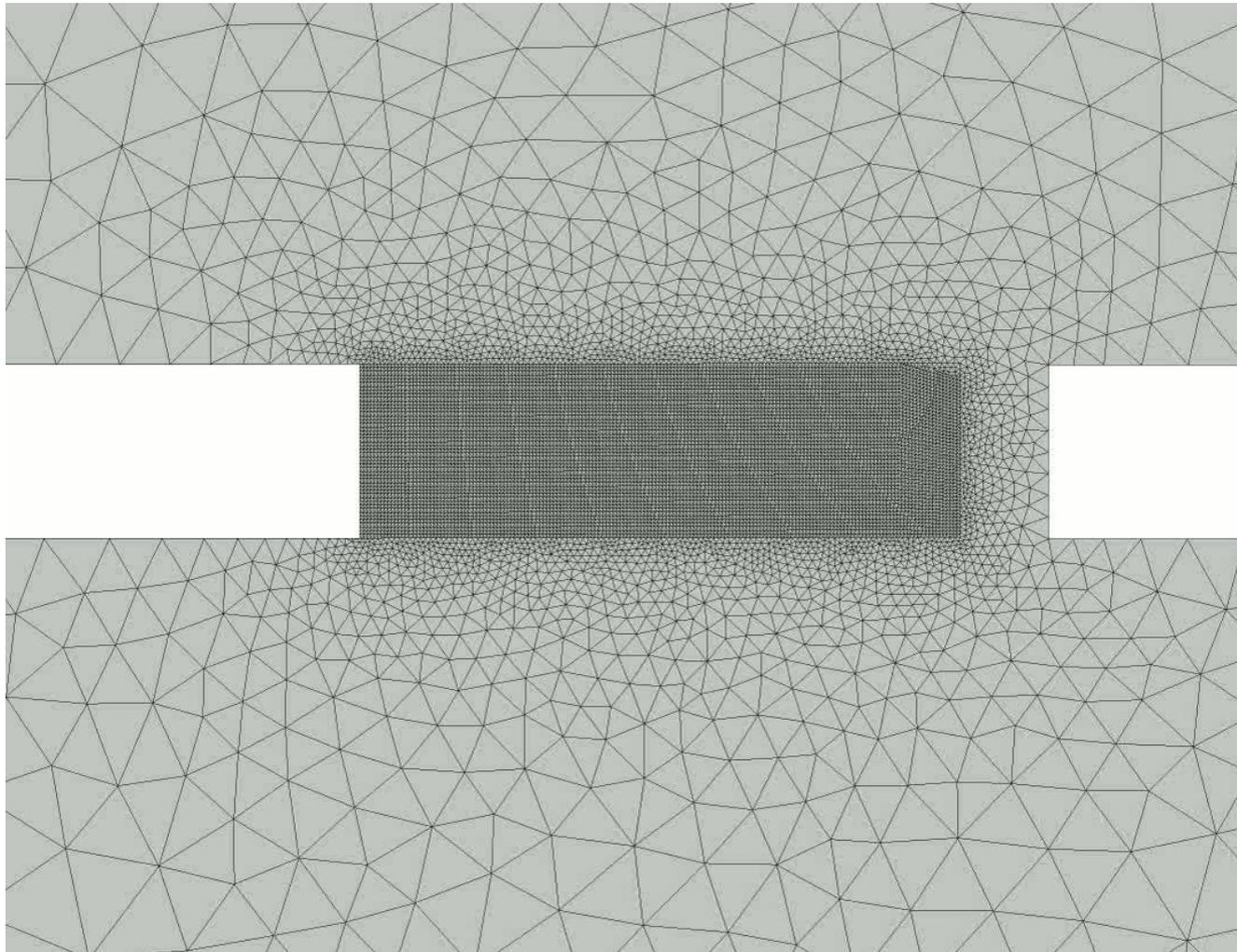
# Mesh size and input velocity order effect



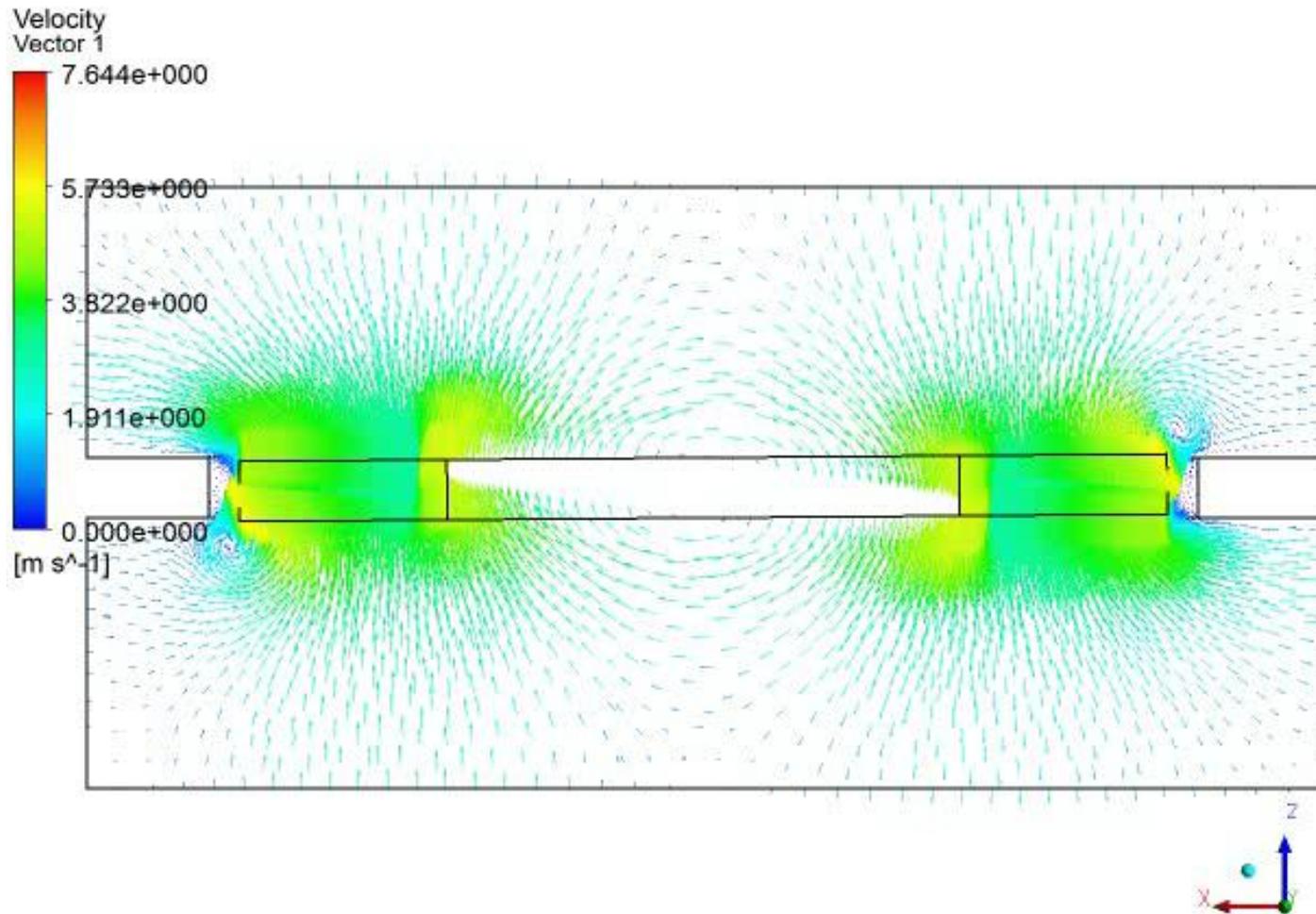
- Fortran subroutine
- Dynamic remeshing procedure
  - If mesh quality decays critically
  - Extracting the new geometry parameters
  - Update the geometry
  - Mesh the new geometry
  - Import the new mesh to the solver
  - Set the initial condition based on the previous solution step
- Mesh quality index:  
Minimum orthgonality angle



- Mesh quality degradation and remeshing process occurs repeatedly

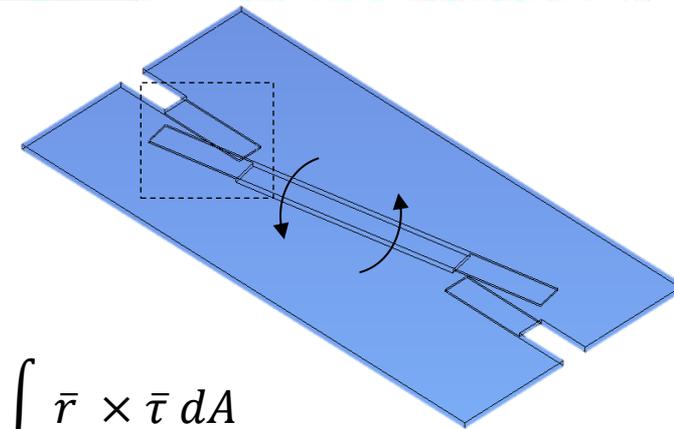
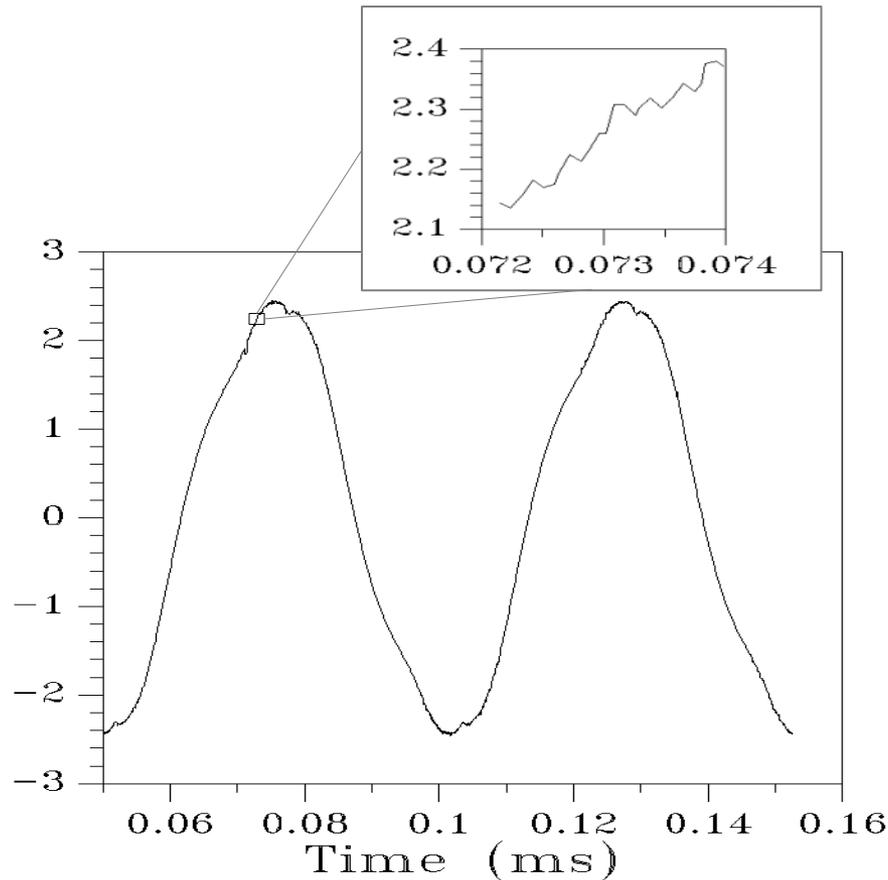
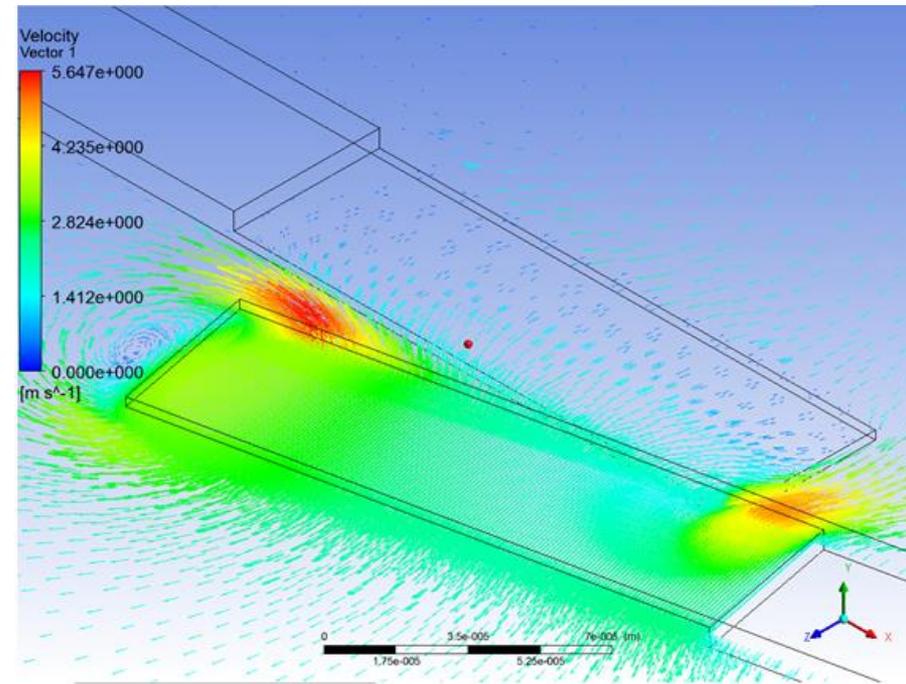


- Air velocity vectors evolving during large oscillations



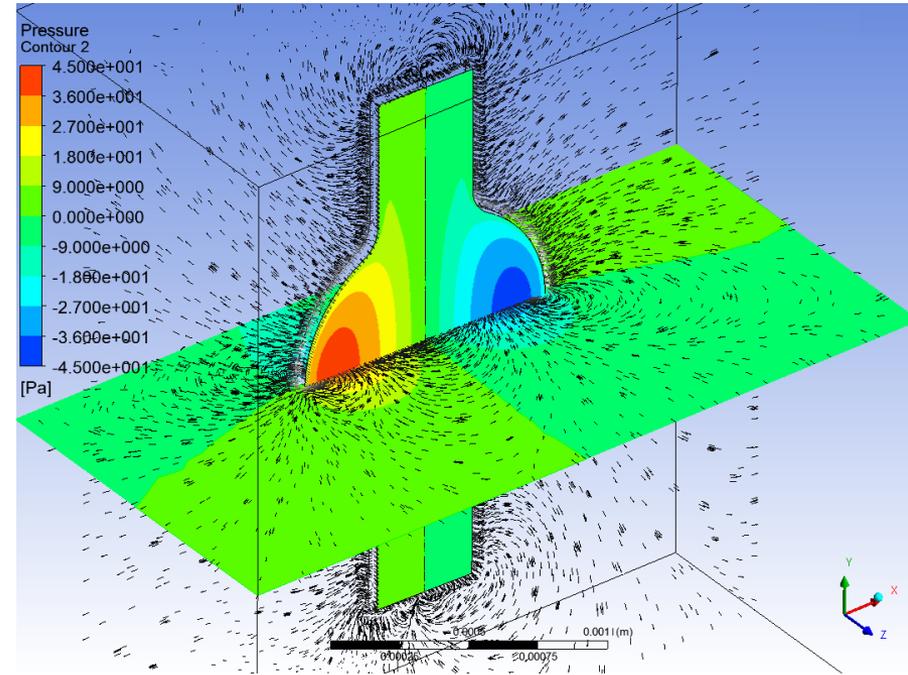
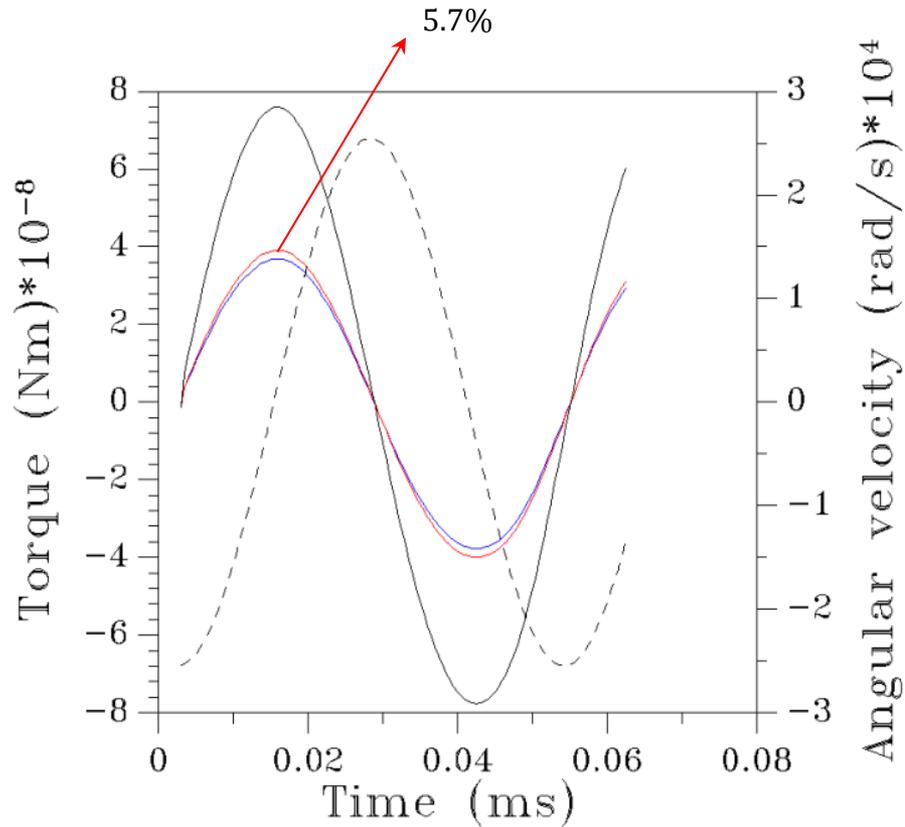
# Rotational- large oscillation

- The finite size of plates results in having complex flow during large oscillations
- Small fluctuations at each remeshing



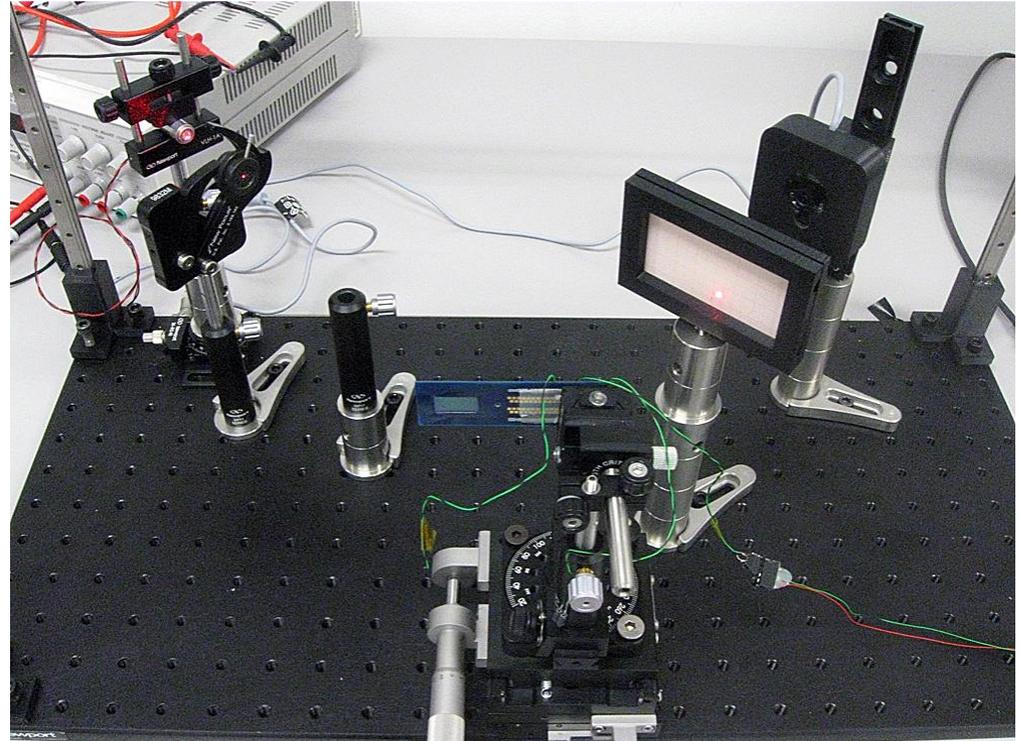
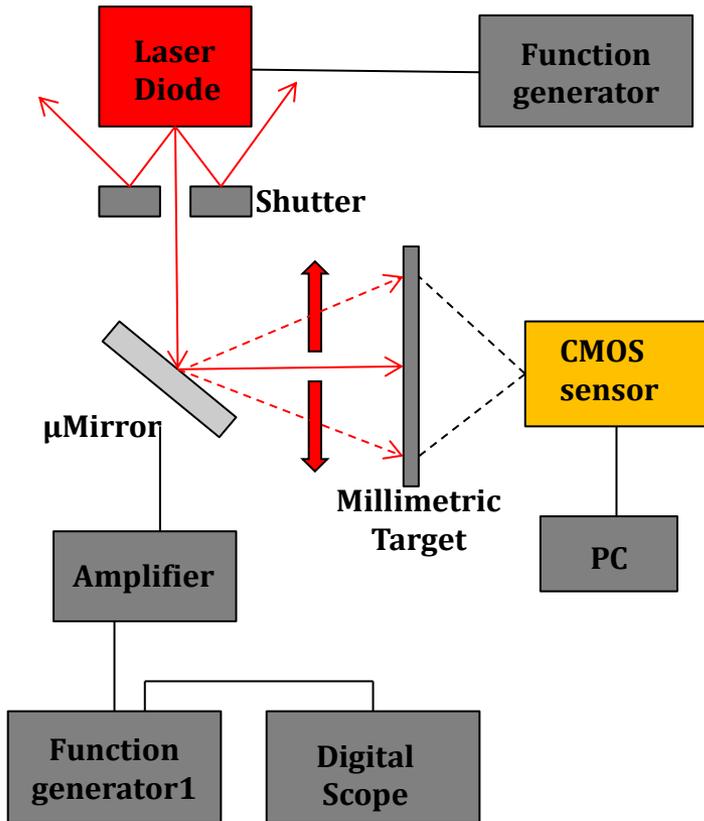
$$\bar{T}_\tau = \int_A \bar{r} \times \bar{\tau} dA$$

→ Negligible squeeze effect



$$\bar{T}_p = \int_A \bar{r} \times p \bar{n} dA$$

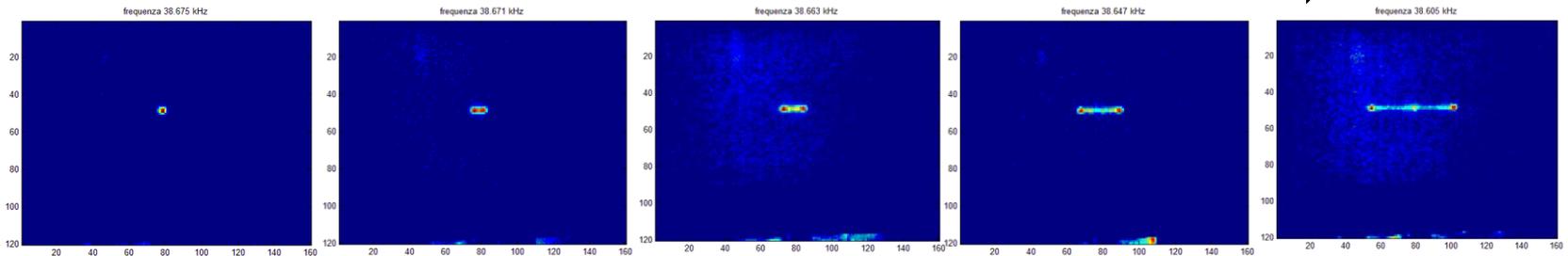
- Total torque ———
- Torque on top ———
- Torque on Bottom ———
- Angular velocity - - - - -



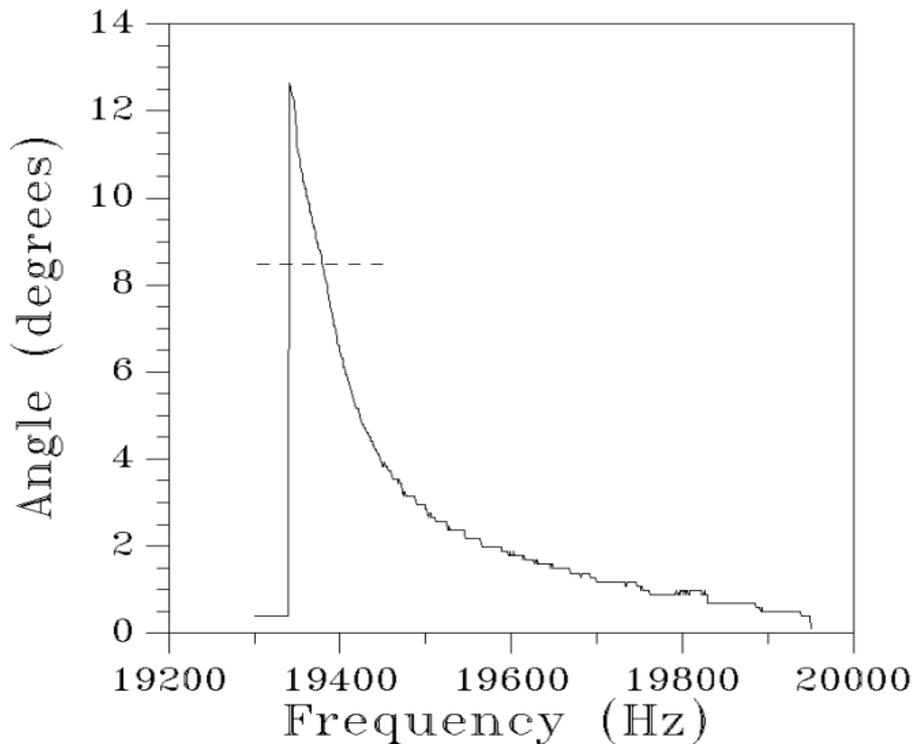
High

Frequency

Low



	$E_{loss}(J)$	$E_{st}(J)$	Quality factor
Comb fingers	1.78e-8	-	1 074
Mirror plate	1.18e-8	-	1 618
Total	2.96e-8	3.04e-6	645



$$Q = 2\pi \frac{E_{st}}{E_{loss}} \quad Q_{total} = \frac{1}{\frac{1}{Q_{comb}} + \frac{1}{Q_{plate}}}$$

$$HBP \rightarrow Q_{exp} = 623$$

→ Model modifications: Including comb fingers at the drag damping and slip BC

- At large oscillations **the end-effect/finite size** has an important role, which contributes in large energy dissipations
- **Individual quality factors** for different damping mechanisms have been obtained and the **overall quality factor** shows good agreement with the **experimental one**
  
- Analysis of the **electrostatic field** at the comb fingers to characterize the system excitation. This would give a complete model for the **dynamics of the micromirror**